

## Question 1A

a) The three conditions for a Couette flow:

- Pressure Gradient is constant
- The flow is uniform
- The flow is steady

b) four conditions that can be used to determine the nature of flow are

- The velocity of flow (m/s)
- The fluid viscosity ( $\text{Ns/m}^2$ )
- The cross sectional diameter of the pipe (m)
- The density of the fluid passing through the pipe.

c) The differences between aerofoil and hydrofoil are enlisted below:

Aerofoil	Hydro foil
1) It is mainly used to lift airplanes and jets	It is mainly used to overcome drag and make machines move with a high velocity in water
2) It is a lifting device mainly used in gaseous fluid (air in particular)	The hydrofoil is a lifting device mainly utilized in liquids fluids (like water)

## Question 1B

## solution

$$\text{Data } \mu = 0.9 \text{ centipoise} = 0.9 \times 10^{-2} \text{ poise} = 0.9 \times 10^{-3} \text{ Ns/m}^2$$

$$u = 1 \text{ m/s}$$

$$b = 10 \text{ mm} = 0.01 \text{ m}$$

$$dp = 60 \text{ kN/m}^2$$

$$dz = 60 \text{ m}$$

$$\text{pressure difference gradient} = \frac{\delta p}{\delta x} = \frac{-60000}{60} = -1 \times 10^3 \text{ N/m}^2$$

~~Q. 100~~

i) Velocity distribution  $u = \frac{Uy}{b} - \frac{1}{2\mu} \left( \frac{\delta p}{\delta x} \right) (by - y^2)$

$$u = 100y + 5555.56y - 55555.56y^2$$

$$u = (5.5556 \times 10^3)y - (5.556 \times 10^5)y^2$$

ii) Discharge per unit width  $= q = \frac{U^2 b}{2} - \frac{b^3}{12\mu} \left( \frac{\delta p}{\delta x} \right)$

$$q = 0.085 + 0.09259$$

$$= 0.09759 \text{ m}^3/\text{s/m}$$

iii) Shear stress at upper plate is @  $y = b$

$$\tau = \frac{\mu U}{b} - \frac{1}{2} \left( \frac{\delta p}{\delta x} \right) (b - 2y)$$

$$\tau = \frac{\mu U}{b} - \frac{1}{2} \left( \frac{\delta p}{\delta x} \right) (-b)$$

$$\tau = 0.09 - 5 = -4.91 \text{ N/m}^2$$

## Question 2

Data

$$\mu = 0.9 \text{ Ns/m}^2$$

$$\rho = 1260 \text{ kg/m}^3$$

$$U = -1.5 \text{ m/s}$$

$$b = 10 \text{ mm} = 0.01 \text{ m}$$

$$P_1 = 250 \text{ kN/m}^2$$

$$P_2 = 80 \text{ kN/m}^2$$

But  $P_1 = P_2 + \rho g z$  (piezometric)

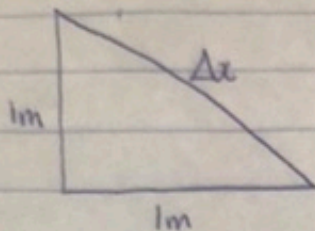
$$250000 + (1260) \times (9.81) \times (1)$$

$$= 262.36 \text{ kN/m}^2$$

and  $P_2 = P_1 + \rho g z$  (piezometric)

$$80000 + (1260) \times (9.81) \times (0)$$

$$80 \text{ kN/m}^2$$



Because the two plates are aligned at an angle of  $45^\circ$ , the above diagram can be used to calculate the change in  $x$ ,

By Pythagoras theorem,

$$\Delta x = \sqrt{1^2 + 1^2} = \sqrt{2} \text{ m}$$

Therefore,

$$\frac{\delta p}{\delta x} = \frac{-(P.1 - P.2)}{\Delta x} = \frac{-(262.36 - 80)}{\sqrt{2}} = -128.948 \text{ kN/m}^2$$

i) Velocity distribution =  $u = \frac{Ux}{b} - \frac{1}{2\mu} \left( \frac{\delta p}{\delta x} \right) (by - y^2)$

$$u = -150y + 716.38y - 71637.8y^2$$

$$u = -(7.16378 \times 10^4)y^2 + 565.62y$$

ii) Shear distribution =  $\tau = \frac{\mu U}{b} - \frac{1}{2} \left( \frac{\delta p}{\delta x} \right) (b - 2y)$

$$\tau = -135 + 644.74 - 128948y$$

$$\tau = 509.74 - (1.289 \times 10^5)y$$

b) Maximum flow velocity

At maximum flow  $\frac{\delta u}{\delta y} = 0$

$$0 = -(1.4328 \times 10^5)y + 565.62$$

$$y = 3.9476 \times 10^{-3} \text{ m}$$

$$U_{\max} = -(7.16378 \times 10^4)(3.9476 \times 10^{-3})^2 + 565.62(3.9476 \times 10^{-3})$$

$$U_{\max} = 1.12 \text{ m/s}$$

c) Shear stress at upper plate  $w$  @  $y = b$

Therefore,  $\tau = 509.74 - (1.289 \times 10^5)y$

$$\tau = 509.74 - (1.289 \times 10^5)(0.01)$$

$$\tau = -779.26 \text{ N/m}^2,$$