

DELINA PRECIOUS

18 LENGOSLOI

Civil Engineering

FLUID MECHANICS (ENG 214)

1) given:

$$L = 2.0m$$

$$V_1 = 5m/s$$

$$P_1/\rho g = 2.5m \text{ of liquid}$$

$$V_2 = 2m/s$$

The loss of head in the tube:

$$= h_c = \frac{0.35(V_1 - V_2)^2}{2g}$$

$$= \frac{0.35[5-2]^2}{2g} = \frac{0.35 \times 9}{2 \times 9.81}$$

$$= 0.16m$$

$$\text{Pressure head} = \frac{P_2}{\rho g}$$

Applying Bernoulli's equation we get:

$$\frac{P_1}{\rho g} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\rho g} + \frac{V_2^2}{2g} + Z_2 + h_c = 18 + 4.08 = 22.08m \text{ of water.}$$

$$= 2208 \text{ cm of water.}$$

Let the datum line pass through the lower end; $Z_2 = 0$, $Z_1 = 2.0$

$$2.5 + \frac{5^2}{2 \times 9.81} + 2.0 = \frac{P_2}{\rho g} + \frac{2^2}{2 \times 9.81} + 0 + 0.16$$

$$2.5 + 1.27 + 2.0 = \frac{P_2}{\rho g} + 0.203 + 0.16$$

$$\frac{P_2}{\rho g} = (2.5 + 1.27 + 2.0) - (0.203 + 0.16)$$

$$= 5.77 - 0.363$$

2) Inlet Diameter: $d_1 = 20cm$
 $A_1 = \frac{\pi \times (20)^2}{4} = 314.16 \text{ cm}^2$

Throat diameter $d_2 = 10cm$

$$A_2 = \frac{\pi \times 10^2}{4} = 78.74 \text{ cm}^2$$

$$P_1 = 17.658 \text{ N/cm}^2$$

$$= 17.658 \times 10^4 \text{ N/m}^2$$

$$\text{density of water} = 1000 \text{ kg/m}^3$$

$$\therefore \frac{P_1}{\rho g} = \frac{17.658 \times 10^4}{9.81 \times 1000} = 18m \text{ of water}$$

$$\frac{P_2}{\rho g} = -30cm \text{ of mercury}$$

$$= -0.30m = -0.30 \times 13.6$$

$$= -4.08m \text{ of water.}$$

∴ Differential head:

$$= h = \frac{P_1}{\rho g} - \frac{P_2}{\rho g} = 18 - (-4.08)$$

The discharge $Q = C_d \frac{A_1 A_2}{\sqrt{A_1^2 - A_2^2}} \times \sqrt{2gh}$

$$= 0.98 \times 314.16 \times 78.54 \times \sqrt{2 \times 9.81 \times 22.08}$$

$$\times \sqrt{(314.16)^2 - (78.74)^2}$$

$$= 16548.3 \text{ cm}^3/\text{s} = 165.48 \text{ lit/s}$$

3) Orifice diameter: $d_o = 15\text{cm}$

$$\therefore \text{Area } a_o = \frac{\pi}{4} (15)^2 = 176.7\text{cm}^2$$

Diameter of pipe $d_1 = 30\text{cm}$

$$\text{Area } a_1 = \frac{\pi}{4} (30)^2 = 706.85\text{cm}^2$$

Specific gravity of oil $S_o = 0.9$

Reading on differential manometer:
 $x = 50\text{cm}$ of mercury

Differential head:

$$h = x \left[\frac{S_g}{S_o} - 1 \right] = 50 \left[\frac{13.6}{0.9} - 1 \right]$$
$$= 50 \times 14.11 = 705.5\text{cm of oil}$$

Where $C_d = 0.64$

$$Q = C_d \cdot \frac{a_o a_1}{\sqrt{a_1^2 - a_o^2}} \times \sqrt{2gh}$$

$$= 0.64 \times \frac{176.7 \times 706.85}{\sqrt{(706.85)^2 - (176.7)^2}} \times \sqrt{2 \times 9.81 \times 705.5}$$

$$= 13741.26\text{cm}^3/\text{s}$$

$$= 137.41\text{ litres/s}$$

4) Diff. of mercury level:

$$x = 170\text{mm} = 0.17\text{m}$$

Sp. gr. of mercury = 13.6

Sp. gr. of sea water = 1.026

$$\text{where } h = x \left[\frac{S_g}{S_o} - 1 \right]$$

$$\therefore 0.17 \left[\frac{13.6}{1.026} - 1 \right] = 2.0834\text{m}$$

$$V = \sqrt{2gh}$$

$$\therefore V = \sqrt{2 \times 9.81 \times 2.0834}$$
$$= 6.393\text{ m/s}$$

$$= \frac{6.393 \times 60 \times 60}{1000}$$

$$= 23.01\text{ km/h}$$

5) Actual flow rate = $0.05\text{m}^3/\text{min}$

m^3/min to m^3/sec

$60\text{secs} \rightarrow 1\text{min}$

$$= \frac{0.05}{60} = 8.33 \times 10^{-4}\text{ m}^3/\text{sec}$$

$$\therefore Q = 8.33 \times 10^{-4}\text{ m}^3/\text{sec}$$

Speed of rotation (N) = 1700 rev/min

$$= \frac{1700}{60} = 28.3\text{ rev/sec}$$

$$= 28.5\text{ rps}$$

$$\begin{aligned} \text{Pressure change} &= 15 \text{ bar} \\ &= 15 \times 10^5 \text{ N/m}^2 \end{aligned}$$

$$\begin{aligned} \text{Shaft Power} &= 15 \times 177.89 \\ &= 2668.35 \text{ watts} \end{aligned}$$

$$\text{Nominal displacement} = 10 \text{ cm}^3/\text{rev}$$

$$1000 \text{ cm} = 1 \text{ m}$$

$$100^3 \text{ cm}^3 = 1 \text{ m}^3$$

$$10 \text{ cm}^3 = x$$

$$\therefore x = \frac{10}{1000000} = 1 \times 10^{-5} \text{ m}^3/\text{rev}$$

iv) Overall Efficiency:

$$= \frac{\text{fluid power}}{\text{Shaft power}} \times 100\%$$

$$= \frac{1249.5}{2668.35} \times 100\%$$

$$= 46.8\%$$

$$\text{Ideal flow rate} = \text{Nominal} \times \text{Speed displacement}$$

$$= 28.3 \times 1 \times 10^{-5}$$

$$= 2.83 \times 10^{-4} \text{ m}^3/\text{sec}$$

$$\text{i) Volumetric Efficiency} = \frac{\text{Actual flow rate}}{\text{Ideal flow rate}} \times 100\%$$

$$= \frac{8.83 \times 10^{-4}}{2.83 \times 10^{-4}} \times 100\%$$

$$= 294.3\%$$

$$\text{ii) } P_f = \text{Actual flow rate (Q)} \times \text{Pressure drop (P)}$$

$$= 8.83 \times 10^{-4} \times 15 \times 10^5$$

$$= 1249.5 \text{ watts or Nm/sec}$$

$$\text{iii) Shaft power} = T \cdot \omega$$

Where; T = Torque input (Nm)

ω = Angular speed (rad/sec)

$$T = 15 \text{ Nm}$$

$$\omega = 2\pi N = 2 \times \frac{22}{7} \times 25.5$$

$$= 177.89 \text{ rad/sec}$$