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Answers



Given that smaller end: $V_2 = 5 \text{ m/s}$
 $\frac{P_1}{\rho} = 2.5 \text{ m}$

Then bigger end: $V_1 = 2 \text{ m/s}$

$$L = Z_1 - Z_2 = 2 \text{ m}$$

$$h_f = \frac{0.85(V_1 - V_2)^2}{2g}$$

$$\text{Considering head loss: } h_f = \frac{0.85(5-2)^2}{2 \times 9.81} = \frac{3.15}{19.62}$$

$$h_f = 0.16056 \approx 0.1606 \text{ m}$$

Applying Bernoulli's equation between the smaller & bigger end

$$\frac{P_1}{\rho} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\rho} + \frac{V_2^2}{2g} + Z_2 + h_f$$

$$\frac{P_2}{\rho} = \frac{P_1}{\rho} + \frac{V_1^2}{2g} - \frac{V_2^2}{2g} + Z_1 - Z_2 - h_f$$

$$= 2.5 + \left(\frac{5^2 - 2^2}{2 \times 9.81} \right) + 2 - 0.1606$$

$$= 2.5 + 1.0703 + 2 - 0.1606$$

$$\frac{P_2}{\rho} = 5.4097 \text{ m}$$



Inlet: $d_1 = 20 \text{ cm} = 20 \times 10^{-2} \text{ m}$

$$A = \frac{\pi d^2}{4} = \frac{\pi \times (20 \times 10^{-2})^2}{4}$$

Throat diameter: $d_2 = 10 \text{ cm} = 10 \times 10^{-2} \text{ m}$

$$A = \frac{\pi d^2}{4} = \frac{\pi \times (10 \times 10^{-2})^2}{4}$$

$$A_1 = 0.0314 \text{ m}^2$$

$$A_2 = 7.85 \times 10^{-3} \text{ m}^2$$

$$P_1 = 17.658 \text{ N/cm}^2 \quad C_d = 0.98$$

$$= 17.668 \times 10^4 \text{ N/m}^2$$

To get h ,

$$\frac{P_1}{\rho} - \frac{P_2}{\rho} = h$$

$$P_1 = 17.658 \times 10^4 \text{ N/m}^2$$

$$\rho = 9.91 \times 10^3 \text{ N/m}^3$$

but we have that throat vacuum pressure = 30 cm of Hg

$$= 0.3 \text{ mHg}$$

$$= 0.3 \times 13.6 = 4.08$$

$$P_2 = -4.08 \text{ (since vacuum pressure)}$$

$$\text{Then } \frac{P_2}{\rho} = \frac{17.658 \times 10^4 \text{ N/m}^2}{9.91 \times 10^3 \text{ N/m}^3} = 18 - 4.08 = 22.07 \text{ m}$$

$$\frac{P_1}{\rho} - \frac{P_2}{\rho} = 18 - (-4.08) = 22.07 \text{ m}$$

$$Q = C_d A_1 A_2 \sqrt{\frac{2gh}{A_1^2 - A_2^2}}$$

$$= 0.98 \times 0.0314 \times 7.85 \times 10^{-3} \times \sqrt{\frac{2 \times 9.81 \times 22.07}{(0.0314^2 - (7.85 \times 10^{-3})^2)}}$$

$$= 2.4166 \times 10^{-4} \times 684.59 = 0.1663$$

$$Q = 0.1663 \text{ m}^3/\text{s}$$

3 Orifice meter: Given that,

$$d_0 = 15 \text{ cm} = 15 \times 10^{-2} \text{ m} \quad \text{Pipe diameter } d_p = 30 \text{ cm} = 30 \times 10^{-2} \text{ m}$$

$$A_0 = \pi \times (15 \times 10^{-2})^2$$

4

$$= 0.07067 \text{ m}^2$$

Sp.g of oil = 0.9

Coefficient of discharge = 0.64

$$\text{Differential head } h = y \left[\frac{5h_L}{50} - 1 \right]$$

$$5h_L = 13.6$$

$$y = 50 \times 10^{-2}$$

$$h = 50 \times 10^{-2} \left[\frac{13.6}{50} - 1 \right]$$

$$h = 50 \times 10^{-2} \times 14.11 = 7.065 \text{ m}$$

$$Q = C_d A$$

$$= 0$$

$$= 3$$

4 Given:

5 Given
Act
Flow

Press

Spee

Non

laden

$$Q = C_d A_o A_p \sqrt{2gh} \sqrt{A_p^2 - A_o^2}$$

$$= 0.64 \times 0.01767 \times 0.07069 \sqrt{2 \times 9.81 \times 0.65} \sqrt{(0.07069^2) - (0.01767^2)}$$

$$= \frac{7.994 \times 10^{-4} \times 11.765}{\sqrt{4.68 \times 10^{-3}}}$$

$$= 0.1374 \text{ m}^3/\text{s}$$

4 Given: Reading of the manometer $y = 170 \text{ mm Hg} = 0.17 \text{ m Hg}$

Sp gravity of mercury $S_{Hc} = 13.6$

Sp gravity of seawater $= 1.026$

$$h = y \left(\frac{S_{Hc}}{S_w} - 1 \right)$$

$$= 0.17 \left(\frac{13.6}{1.026} - 1 \right)$$

$$h = 0.17 \times 12.255 = 2.083 \text{ m}$$

$$V = \sqrt{2gh} = \sqrt{2 \times 9.81 \times 2.083} = 6.39 \text{ m/s}$$

5 Given that:

Actual Flow rate = $600 \text{ m}^3/\text{min}$

$$= \frac{600 \text{ m}^3}{60 \text{ s}} = 10 \text{ m}^3/\text{s}$$

Pressure = $15 \text{ bar} = 15 \times 10^5 \text{ N/m}^2$

$$\text{Speed} = \frac{1700 \text{ rev/min}}{60} = 28.33 \text{ rev/s}$$

Nominal displacement = $10 \text{ cm}^3/\text{rev}$

$$= 10^{-5} \text{ m}^3/\text{rev}$$

Ideal flow rate = Nominal \times Speed displacement

$$28.33 \times 10^{-5}$$

$$= 2.833 \times 10^{-4} \text{ m}^3/\text{sec}$$

$$\begin{aligned} \text{Volumetric Efficiency} &= \frac{\text{Actual Flowrate}}{\text{Ideal Flowrate}} \times 100\% \\ &= \frac{8.33 \times 10^{-5} \text{ m}^3/\text{s}}{2.833 \times 10^{-4} \text{ m}^3/\text{s}} \times 100\% \\ &= \underline{\underline{29.41\%}} \end{aligned}$$

$$\begin{aligned} \text{Fluid Power} &= Q \cdot \Delta P \\ &= 8.33 \times 10^{-4} \times 15 \times 10^5 \\ &= 1249.5 \text{ Nm/sec} \end{aligned}$$

$$\text{Shaft Power} = T \cdot \omega$$

where T = Torque input (Nm)

ω = Angular speed

$$T = 15 \text{ Nm}$$

$$\omega = 2\pi N \text{ for rpm} \quad \omega = \frac{2\pi N}{60} \quad 2 \times \pi \times 283$$

$$\omega = 177.8 \text{ rad/sec}$$

$$\omega = \frac{2 \times 2\pi \times 15}{60}$$

$$\text{Shaft Power} = 15 \times 177.8 = 2667.2 \text{ Watts}$$

$$\text{Overall Efficiency} = \frac{\text{Fluid Power}}{\text{Shaft Power}} \times 100$$

$$= \frac{1260}{2667.2} \times 100 = \underline{\underline{46.87\%}}$$