

JOHN-OGBE JONATHAN
19/ENG04/062
ELECTRICAL ENGINEERING

1.

Let the smaller end be represented by ①
 The larger end be represented by ②

Solution

$l = 2.0m$ $P_1 / \rho g = 2.5m$
 $V_1 = 5m/s$ $V_2 = 2m/s$

Loss of head = $h_L = \frac{0.35(V_1 - V_2)^2}{2g} = \frac{0.35(5-2)^2}{2g} = \frac{0.35(3)^2}{2 \times 9.81}$

$= \frac{0.35(9)}{2 \times 9.81}$

pressure head = $P_2 / \rho g = 2cm$

Applying Bernoulli's eq. at ① & ②

$\frac{P_1}{\rho g} + \frac{v_1^2}{2g} + z_1 = \frac{P_2}{\rho g} + \frac{v_2^2}{2g} + z_2 + h_L$

$z_2 = 0, z_1 = 2.0$

$\therefore 2.5 + \frac{(5)^2}{2 \times 9.81} + 2.0 = \frac{P_2}{\rho g} + \frac{(3)^2}{2 \times 9.81} + 0 + 0.016$

$2.5 + 1.27 + 2.0 = \frac{P_2}{\rho g} + 0.205 + 0.016$

$\frac{P_2}{\rho g} = (2.5 + 1.27 + 2.0) - 0.221$

$= 5.77 - 0.363 = 5.407$

$= 5.4m$ of fluid.

2 Inlet Diameter (D_1) = ~~20~~²⁰ cm = 0.2 m

throat
Outlet Diameter (D_2) = 10 cm = 0.1 m

$$\text{Area of Inlet} = \frac{\pi}{4} \times (0.2)^2 = 0.031416 \text{ m}^2$$

$$\text{Area of throat} = \frac{\pi}{4} \times (d_2)^2 = \frac{\pi}{4} (0.1)^2 = 0.007854 \text{ m}^2$$

$C_d = 0.98$, pressure (P_1) = $17.653 \times 10^4 \text{ N/m}^2$, $P_2 = 100000 \text{ Pa}$

$$P_1 / \rho_w \cdot C_d = \frac{17.653 \times 10^4}{9.81 \times 1000} = 18 \text{ m}$$

$P_2 / \rho_w \cdot C_d = 30 \text{ cm}$ of mercury = $0.3 \times 13.6 = 4.08 \text{ m}$
differential head.

$$h = P_1 / \rho_w \cdot C_d - P_2 / \rho_w \cdot C_d = 18 - 4.08 = 13.92 \text{ m water}$$

$$Q_1 = C_d \times a_1 \cdot a_2 \frac{\sqrt{2gh}}{\sqrt{(a_1)^2 - (a_2)^2}}$$

$$= 0.98 \times 314.16 \times 0.007854 \times \frac{\sqrt{2 \times 9.81 \times 13.92}}{\sqrt{(314.16)^2 - (0.007854)^2}}$$

$$= \frac{50328837.21 \times 165555}{304} = 0.165 \text{ m}$$

$$= 165.56 \text{ lit/s}$$

3 Orifice Diameter = 15 cm

Pipe Diameter = 30 cm

Co-efficient of discharge of the orifice is 0.64

flow of oil of specific gravity = 0.9

Soln

$$A_o = \pi/4 (15)^2 = 176.714 \text{ cm}^2 \text{ (A., Area of the orifice)}$$

$$A_p = \pi/4 (30)^2 = 706.858 \text{ cm}^2 \text{ (Area of the pipe)}$$

$$H = \left[\frac{13.6}{0.9} - 1 \right] + 50 \text{ cm of oil}$$

$$= [15.1 - 1] \times 50 \text{ cm} = 14.1 \times 50$$
$$= 705.56$$

$$Q_1 = \frac{C_d A_o A_p \sqrt{2gh}}{\sqrt{(A_p^2) - (A_o)^2}}$$

$$Q = \frac{0.64 \times 176.71 \times 706.86 \times \sqrt{2 \times 9.81 \times 7.05 \times 100}}{\sqrt{(706.85)^2 - (176.74)^2}}$$

$$Q = 137414.25 \text{ cm}^3/\text{sec}$$

$$\text{Litres} = 137.41425 \text{ Lit/sec}$$

$$\text{Rate of flow of oil} = 137.414 \text{ Lit/sec}$$

Diff of Mercury level $x = 170\text{mm} = 0.17\text{m}$

Sp gr of mercury = 13.6

Specific gravity (sp) = 1.026

Soln

$$H = x \left[\frac{\text{sg}}{\text{sp}} - 1 \right] = 0.17 \left[\frac{13.6}{1.026} - 1 \right]$$
$$= 2.0834\text{m}$$

$$v = \sqrt{2gh}$$

$$= 2 \times 9.81 \times 2.08$$

$$= 6.39\text{m/s}$$

$$= 6.39 \times 60 \times 60$$

1000

$$= \frac{23004}{1000} = 23.004$$

$$\text{Speed of submarine} = 23.004\text{km/hr}$$

$$= 23.004\text{ms/hr}$$

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Rate of Pump - $0.05 \text{ m}^3/\text{min} = 500 \text{ cm}^3/\text{min}$

Pressure charge - 15 bar

Speed rotation - 1700

normal displacement - $10 \text{ cm}^3/\text{rev}$

tongue input = 15 N/m

Solo

Heat flow rate \rightarrow Normal displacement \times speed
 $= 15 \times 1700 = 25,500 \text{ cm}^3/\text{min}$
 $= 25.500 \text{ dm}^3/\text{min}$

Volumetric efficiency

$$= \frac{\text{Actual flow}}{\text{ideal flow}} = \frac{500}{25.5} = 19.60$$

$$Q = \frac{500 \times 10^{-3}}{60 \text{ m}^3/\text{s}} = 83.3 \times 10^{-4} \text{ m}^3/\text{s}$$

Fluid Power

$$\Delta P = 100 \times 10^5 \text{ N/m}^2$$

$$\text{Fluid Power} = \Delta P = 83.3 \times 10^{-4} \times 100 \times 10^5$$

$$\text{Fluid Power} = 83300 \text{ W}$$

Shaft Power =

$$= 2\pi \text{ N} / 60 = 2\pi \times 1700 \times 25.5 = 4541.4 \text{ W/m}$$

Overall Efficiency 60

$$= \frac{\text{Fluid Power}}{\text{Shaft Power}} = \frac{83300}{4541.4} = 18.342$$

$$O.E = 18.342 \text{ or } 1834.2\%$$