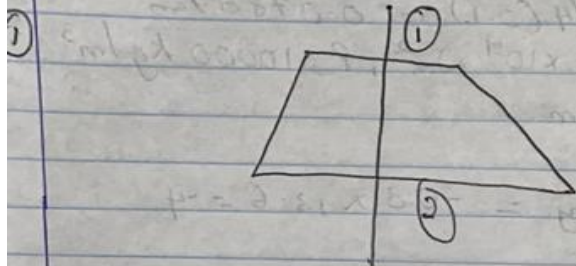


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Mechanical Engineering

Eng 2



Let smaller end be represented by ①

lower end be represented by ②

Solution

$$L = 2.0 \text{ m}$$

$$P_1, S_{pg} = 2.5 \text{ m}$$

$$V_1 = 5 \text{ m/s}$$

$$V_2 = 2 \text{ m/s}$$

$$\text{loss of head} = h_L = 0.35 \frac{(V_1 - V_2)^2}{2g} = \frac{0.35 (5 - 2)^2}{2 \times 9.81} = \frac{0.35 (3)^2}{2 \times 9.81}$$
$$= 0.35 (9)$$

$$\text{pressure head} = \frac{P_2}{\rho g} = 7 \text{ cm}$$

Applying Bernoulli's eq at ① and ②

$$\frac{P_1}{\rho g} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\rho g} + \frac{V_2^2}{2g} + Z_2 + h_L$$

$$Z_2 = 0.2, Z_1 = 2.0$$

$$\therefore \frac{2.5}{2 \times 9.81} + \frac{(5)^2}{2 \times 9.81} + 2.0 = \frac{P_2}{\rho g} + \frac{(2)^2}{2 \times 9.81} + 0.016$$

$$2.5 + 1.27 + 2.0 = \frac{P_2}{\rho g} + 0.203 + 0.16$$

$$\frac{P_2}{\rho g} = (2.5 + 1.27 + 2.0) - (0.203 + 0.16)$$

$$= 5.77 - 0.363 = 5.407$$

$$= 5.4 \text{ m of fluid.}$$

②

DATA

Inlet Diameter (D_1) = 20cm = 0.2m

throat Diameter (D_2) = 10cm = 0.10m

Area of Inlet = $\frac{\pi}{4} \times (0.2)^2 = 0.031416 \text{ m}^2$

Area of throat = $\frac{\pi}{4} \times (0.1)^2 = 0.007854 \text{ m}^2$

$C_d = 0.98$, pressure (P_1) = $17.685 \times 10^4 \text{ N/m}^2$, $P_2 = 10000 \text{ kg/m}^3$

$$P_1/\rho_1 \text{ e.g.} = \frac{17.685 \times 10^4}{9.81 \times 1000} = 18 \text{ m}$$

$$P_2/\rho_2 \text{ e.g.} = 30 \text{ cm of mercury} = -0.3 \times 13.6 = -4$$

differential head.

$$h = P_1/\rho_1 - P_2/\rho_2 = 18 - (-4) = 22.08 \text{ m water}$$

$$Q = \frac{C_d \times a_1 a_2 \sqrt{2gh}}{\sqrt{(a_1)^2 - (a_2)^2}}$$

$$= \frac{0.98 \times 314.16 \times 0.007854 \times \sqrt{2 \times 9.81 \times 22.08}}{\sqrt{(314.16)^2 - (0.007854)^2}}$$

$$= \frac{50328837.21 \times 165555}{304} = 0.165 \text{ m}$$

304

$$= 165.561 \text{ m/s}$$

③

DATA

Orifice Diameter = 15cm

Pipe Diameter = 30cm

Coefficient of discharge of the meter is 0.64

flow of oil of specific gravity = 0.9.

Solution

$a_1 = \frac{\pi}{4} \times (30)^2 = 706.86 \text{ cm}^2$ (Area of the pipe)

$$Q_1 = \frac{C_d A_0 A_p \sqrt{2gh}}{\sqrt{(A_p^2) - (A_0)^2}}$$

$$Q = \frac{0.64 \times 176.71 \times 706.86 \times \sqrt{2 \times 9.81 \times 7.05 \times 100}}{\sqrt{(706.85)^2 - (176.74)^2}}$$

$$Q = 137414.25 \text{ cm}^3/\text{sec}$$

$$\text{Litres} = 137.41425 \text{ lit/sec}$$

$$\text{Rate of flow of } 0^\circ = 137.414 \text{ lit/sec}$$

$$(4) \text{ manometer reading } (y) = 170 \text{ mm Hg} = 0.17 \text{ m Hg}$$

$$S_g h = 13.6$$

$$S_g \text{ of sea water } (s_w) = 1.026$$

$$h = 0.17 \left[\frac{13.6}{1.026} - 1 \right]$$

$$\text{known } = 2.08$$

Velocity of the submarine

$$v = \sqrt{2gh}$$

$$= \sqrt{2 \times 9.81 \times 2.08}$$

$$= 0.39 \text{ m/s}$$

5. Q = solution

$$= \left[\frac{5}{1000} \times \frac{1}{60} \right] \text{ m}^3/\text{s}$$

$$= 8.3 \times 10^{-5} \text{ m}^3/\text{s}$$

$$v = 1700 \text{ rev/min} = \frac{1700}{60}$$

$$= 28.33 \text{ rev/sec}$$

$$Q_p = 10 \text{ cm}^3/\text{rev} = 10 \times 10^{-6} \text{ m}^3/\text{rev}$$

Ideal flow rate = speed \times displacement

$$1 \times 10^{-5} \times 28.33$$

$$= 2.833 \times 10^{-4} \text{ m}^3/\text{sec}$$

$$i) \text{ Vol. efficiency} = \frac{\text{actual } Q}{\text{ideal } Q} \times 100$$

$$= \frac{8.3 \times 10^{-5}}{2.83 \times 10^{-4}} \times 100 = 29.33\%$$

$$ii) \text{ fluid power} = \text{flow rate} \times \text{pressure}$$

$$= 8.3 \times 10^{-5} \times (15 \times 10^5)$$

$$= 124.5 \text{ W}$$

$$iii) \text{ shaft power} = \text{torque} \times \text{angular velocity}$$

$$\omega = 2\pi N$$

$$= 2\pi \times 28.33 = 178 \text{ rad/s}$$

$$\text{shaft power} = 15 \times 178 = 2670 \text{ W}$$

$$= 2670 \text{ W}$$

$$\text{Overall efficiency} = \frac{\text{fluid power}}{\text{shaft power}} \times 100$$

$$= \frac{124.5}{2670} \times 100$$

$$= 4.66\%$$

$$= 4.66\%$$

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