

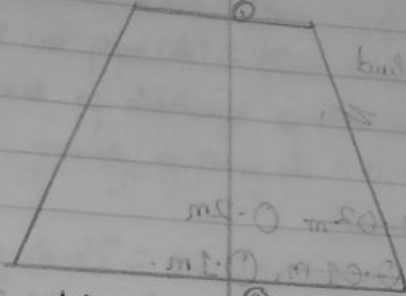
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MATRIC NUMBER: 28 (ENG 08 022)

DEPARTMENT: BIOMEDICAL ENGINEERING

DATE: APRIL, 2020

COURSE: FLUID MECHANICS (ENG 21P)



Let smaller end be represented by (1), and larger end be represented by (2)

$l = 2.0m$, $v_1 = 5m/s$, $v_2 = 2m/s$, $P_1/\rho g = 2.5m$ of the fluid liquid.

① Loss of head at the tube.

$$h_L = \frac{0.35}{2g} (v_1 - v_2)^2$$

$$= \frac{0.35}{2 \times 9.81} (5 - 2)^2 = \frac{0.35 \times 9}{19.62} = 1.62$$

Pressure head = $\frac{P_1}{\rho g}$

Applying Bernoulli's equation

$$\frac{P_1}{\rho g} + \frac{v_1^2}{2g} + z_1 = \frac{P_2}{\rho g} + \frac{v_2^2}{2g} + z_2 + h_L$$

Let the

$$2.5 + \frac{(5)^2}{2 \times 9.81} + 2.0 = \frac{P_2}{\rho g} + \frac{(2)^2}{2 \times 9.81} + 0 + 0.16$$

$$2.5 + \frac{25}{19.62} + 2.0 = \frac{P_2}{\rho g} + \frac{4}{19.62} + 0.16$$

$$2.5 + 1.27 + 2.0 = \frac{P_2}{\rho_f} + 0.203 \times 10^3 \times 16$$

$$5.77 = \frac{P_2}{\rho_f} + 0.803$$

$$\frac{P_2}{\rho_f} = 5.77 - 0.363$$

$$\frac{P_2}{\rho_f} = 5.407 \text{ m of fluid}$$

2) Inlet diameter (d₁) = 80 cm = 0.8 m, 0.2 m

Exit Throat diameter (d₂) = 10 cm = 0.1 m, 0.1 m.

Area of inlet = $\frac{\pi}{4} \times (0.8)^2 = 0.5024 \text{ m}^2$

Area of throat = $\frac{\pi}{4} \times (0.1)^2 = 7.85 \times 10^{-3} \text{ m}^2$

Cd = 0.98, Pressure (P₁) = 17.685 × 10³ N/m², P = 1600 kg/m³

$$\frac{P_1}{\rho_f} = \frac{17.685 \times 10^3}{1600} = 11.053 \text{ m}$$

$$\frac{P_2}{\rho_f} = 80 \text{ cm of mercury} = -0.5 \times 13.6 = -40 \text{ cm}$$

Differential head

$$h_d = \frac{P_1}{\rho_f} - \frac{P_2}{\rho_f} = 11.053 - (-40) = 51.053 \text{ m of water}$$

$$Q = \frac{C_d \times d_1 a_1 \sqrt{2gh}}{\sqrt{(a_1)^2 - (a_2)^2}}$$

$$Q = \frac{0.98 \times 0.0314 \times 1.75 \times 10^{-3} \times \sqrt{2 \times 9.81 \times 51.053}}{\sqrt{(0.0314)^2 - (1.85 \times 10^{-3})^2}}$$

$$Q \approx \frac{2.415602 \times 10^{-4} \times \sqrt{103.3 \times 109.6}}{\sqrt{0.001 - 3.42 \times 10^{-6}}}$$

$$Q = \frac{2.415602 \times 10^{-4} \times 20.8136878}{\sqrt{9.243375 \times 10^{-4}}}$$

$$Q = 5.02171 \times 10^{-5}$$

$$0.0304$$

$$Q = 0.165 \text{ m}^3/\text{s}$$

$$Q = 165.56 \text{ l/s}$$

88.

Orifice diameter $(d) = 15 \text{ cm} = 0.15 \text{ m}$

Radius of pipe $(R) = 30 \text{ cm} = 0.3 \text{ m}$

Co-efficient of discharge of the orifice $= 0.61$

Specific gravity of flow of oil $= 0.9$

$$A_0 = \frac{\pi}{4} (0.15)^2 = 0.01767 \text{ m}^2$$

$$A_1 = \frac{\pi}{4} (0.30)^2 = 0.07068 \text{ m}^2$$

$$h = \left(\frac{13.6 - 1}{0.1} \right) \times 0.5 \text{ m}$$

$$= (15.1 - 1) \times 0.5 \text{ m}$$

$$= 14.1 \times 0.5 \text{ m}$$

$$= 7.05 \text{ m}$$

$$Q = \frac{C_d A_0 A_1 \sqrt{2gh}}{\sqrt{A_1^2 - A_0^2}}$$

$$Q = \frac{0.61 \times 0.01767 \times 0.07068 \times \sqrt{2 \times 9.81 \times 7.05}}{\sqrt{(0.07068)^2 - (0.01767)^2}}$$

$$Q = \frac{1.993 \times 10^{-4} \times \sqrt{138.3214}}{\sqrt{0.49476 \times 10^{-2} - 0.3122 \times 10^{-4}}}$$

$$Q = \frac{1.993 \times 10^{-4} \times 11.76}{\sqrt{4.9164 \times 10^{-2}}}$$

$$Q = \frac{2.334 \times 10^{-3}}{0.2219}$$

$$Q = 0.0105 \text{ m}^3/\text{s}$$

$$Q = 10.5 \text{ l/s}$$

89. Diff of mercury level $(x) = 170 \text{ mm} = 0.17 \text{ m}$

Specific gravity of sea water $= 1.026$

Specific gravity of mercury $= 13.6$

$$h = \left(\frac{13.6}{1.026} - 1 \right) \times 0.17$$

$$= (13.255 - 1) \times 0.17$$

$$= 2.083 \text{ m} \approx 2.1 \text{ m}$$

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$$v = \sqrt{2gh}$$

$$= \sqrt{2 \times 9.81 \times 2.083}$$

$$= \sqrt{40.87658}$$

$$v = 6.393 \text{ m/s}$$

$$v (\text{km/hr}) = \frac{6.393 \times 60 \times 60}{1000}$$

$$= 23.5094$$

$$= 23.51$$

\therefore Speed of submarine = 23.51 km/hr

5) Actual flow rate $= 5 \text{ dm}^3/\text{min} = 8.335 \times 10^{-5} \text{ m}^3/\text{sec}$

Speed rotation $(N) = 1700 \text{ rev}/\text{min} = 28.3 \text{ rev}/\text{sec}$

Maximal Displacement $= 10 \text{ cm}^3/\text{rev} = 1 \times 10^{-5} \text{ m}^3/\text{rev}$

Torque input $= 15 \text{ Nm}$

Pressure change $= 15 \text{ bar} = 15 \times 10^5 \text{ N/m}^2$

(i) Ideal flow rate $= \text{Normal displacement} \times \text{speed rotation}$

$= 1 \times 10^{-5} \times 28.3$

$= 2.83 \times 10^{-4} \text{ m}^3/\text{sec}$

(ii) Volumetric efficiency $= \frac{\text{Actual flow rate}}{\text{Ideal flow rate}} \times 100\%$

$= \frac{8.33 \times 10^{-5}}{2.83 \times 10^{-4}} \times 100\%$

$= 29.43\%$

$= 29.4\%$

(iii) Fluid Power $= \text{Actual flow rate} \times \text{Pressure change}$

$= 8.335 \times 10^{-5} \times 15 \times 10^5$

$= 124.95 \text{ Nm/sec}$

(iv) Shaft Power $= \text{Torque input} \times \text{Angular speed}$

Angular speed $\Rightarrow 2 \times \pi \times \text{speed of rotation}$

$= 2 \times \pi \times 28.3$

$= 177.8 \text{ rev}/\text{sec}$

Shaft power $= 15 \times 177.8$

$= 2667.2 \text{ Nm/sec}$

(v) Overall efficiency $= \frac{\text{Fluid Power}}{\text{Shaft Power}} \times 100\%$

$= \frac{124.95}{2667.2} \times 100\%$

$= 4.68\%$

$= 4.68\%$

$= 4.68\%$