

ANYANG JUSTWA

15/ENG02/012

MECHANICAL ENGINEERING

PIPELINE ENG.

Question 1

$$Q = 2000 \text{ m}^3/\text{hr} = 33.3 \text{ m}^3/\text{hr}$$

$$P = 1.2 \text{ N/mm}^2$$

$$V = 28 \text{ m/s}$$

$$\sigma_t = 40 \text{ MPa}$$

$$D = ? = 1.13 \sqrt{\frac{Q}{R}} = 1.13 \sqrt{\frac{33.3}{1680}}$$

$$= 0.159 = 160 \text{ mm}$$

Wall thickness

from table 8.2 $C = 3 \text{ mm}$

$$\therefore t = \frac{pD}{2} + C$$

$$t = \frac{(1.2 \times 160)}{2} + 3 = 5.4 \text{ mm} \approx 6 \text{ mm}$$

Question 2

$$D = 200 \text{ mm} \quad P = 0.7 \text{ N/mm}^2$$

from table 8.1 $\sigma_t = 14 \text{ N/mm}^2$

$$8.2 \quad C = 9 \text{ mm}$$

$$t = \frac{PD}{2\sigma_t} + C = \frac{(0.7 \times 200)}{(2 \times 14)} + 9 = 14 \text{ mm}$$

Using the thickness

i) Nominal diameter of bolts

Nominal diameter of bolts.

$$d = 0.75t + 10$$

$$= (0.75 \times 14) + 10 = 20.5 \text{ mm}$$

Number of bolts

$$n = 0.0275 D + 1.6$$

$$= 0.0275 \times 700 + 1.6 = 7.1 \text{ say } 8 \text{ mm}$$

Thickness of the flanges

$$t_f = 1.5t + 3 = (1.5 \times 14) + 3$$

$$= 24 \text{ mm}$$

Width of the flanges

$$W = 2.3d = 2.3 \times 21 = 48.3 \text{ mm}$$

Outside diameter of the bolts

$$D_o = D + 2t + 2R = 200 + 2(14) + 2(50) = 328 \text{ mm}$$

Pitch circle diameter of the holes

$$D_p = D + 2t + 2d + 12$$

$$= 200 + 2(14) + 2(50) = 328 \text{ mm}$$

Pitch circle diameter of the bolts

$$D_p = D + 2t + 2d + 12$$

$$= 200 + 2 \times 14 + 2 \times 21 + 12 = 282 \text{ mm}$$

Cr. Pitch

$$= P_c = \frac{\pi \times D_p}{n} = 110.9 \text{ mm}$$

$$\therefore \text{Since } d_1 = 21 + 3 = 24 \text{ mm}$$

$$21 + 3 = 24 \text{ mm}$$

$$20\sqrt{d_1} = 20\sqrt{24} = 97.9$$

$$30\sqrt{d_1} = 30\sqrt{24} = 140.9$$

$P_c > 20\sqrt{d_1} \therefore$ design is satisfactory

3) Given $D = 50 \text{ mm}$ or $r = 25 \text{ mm}$
 $p = 7 \text{ N/mm}^2$ $\sigma_c = 21 \text{ MPa}$ $\sigma_{br} = 28 \text{ MPa}$

$$t = R \left[\sqrt{\frac{\sigma_c + p}{\sigma_c - p}} - 1 \right] = 25 \left[\sqrt{\frac{21+7}{21-7}} - 1 \right] = 10.35$$

Outer diameter

$$D_i = D + 2 \times 10.35$$

$$= 50 + 2 \times 10.35 = 70.7 \text{ mm}$$

∴ Force trying to separate the flanges

$$F = \frac{\pi (D_i)^2}{4} p = \frac{\pi (70.7)^2}{4} \times 7 = 26943 \text{ N}$$

∴ Since the flange is secured at one end

$$F_b = \frac{F}{2} = \frac{26943}{2} = 13471.5 \text{ N}$$

Let $d_c =$ core diameter

$$13471.5 = \frac{\pi d_c^2}{4} \sigma_{ts} = \frac{\pi d_c^2}{4} \times 28$$

$$\int d_c^2 = \frac{13471.5 \times 4}{28} = 1927.14$$

$$d_c = \sqrt{1927.14} = 43.9 \text{ mm}$$

$$d_c = 24.7 \text{ mm} \approx 25 \text{ mm}$$

* Nominal diameter of bolts

$$d = \frac{d_c}{0.84} = \frac{25}{0.84} = 29.4 \approx 30 \text{ mm}$$

* Outer diameter of the flange

$$D_o = D + 2t + 4.6d$$

$$= 50 + 2(11) + 4.6(30) = 71.2 \text{ mm}$$

From Circle diameter

$$\begin{aligned} D_p &= D_o - (3t + 20\text{mm}) \\ &= 210 - (3 \times 11 + 20) \\ &= 157\text{mm} \end{aligned}$$

From the diagram

$$b = 111.40\text{mm}$$

$$a = 48\text{mm}$$

∴ Bending moment of section x-x

$$\begin{aligned} F_b \times e &= 13471.5 \times 48\text{mm} \\ &= 646632\text{N}\cdot\text{mm} \end{aligned}$$

and section modulus

$$= \frac{1}{6} b (t_f)^2 = \frac{1}{6} \times 111.40 (t_f)^2$$

$$= 18.57 (t_f)^2$$

$$M_{xx} = F_b \times Z$$

$$646632 = 21 \times 18.56 t_f^2$$

$$t_f^2 = \frac{646632}{389.76}$$

$$t_f = \sqrt{1659.05}$$

$$t_f = 40.7 \text{ mm}$$