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1. A pump delivers at the rate of  $10 \text{ dm}^3/\text{min}$  with a pressure change of 12 bar. The speed of rotation is 1500 rev/min while the nominal displacement is given as  $10 \text{ cm}^3/\text{rev}$ . If the torque input is 12.5 Nm.

Compute

- Volumetric efficiency
- Fluid Power
- Shaft Power
- Overall Efficiency.

Soln

$$Q_{\text{actual}} = 10 \text{ dm}^3/\text{min}$$

$$\frac{10 \text{ dm}^3}{1 \text{ min}} \times \frac{1 \text{ m}^3}{1000 \text{ dm}^3} \times \frac{1 \text{ min}}{60 \text{ sec}}$$

$$= \frac{1}{6000} \text{ m}^3 \text{ s}^{-1}$$

$$\Delta P = 12 \text{ bar} = 12 \times 10^5 \text{ Nm}^{-2}$$

$$\text{Speed } \omega = 1500 \text{ rev/min}$$

$$\frac{1500 \text{ rev}}{\text{min}} \times \frac{1 \text{ min}}{60 \text{ sec}}$$

$$= 25 \text{ rev/sec}$$

$$\text{Nominal Displacement} = 10 \text{ cm}^3/\text{rev}$$

$$\frac{10 \text{ cm}^3}{\text{rev}} \times \frac{1 \text{ m}^3}{10^6 \text{ cm}^3} = 10^{-5} \text{ m}^3/\text{rev}$$

$$\text{Torque input } (T) = 12.5 \text{ Nm}$$

$$\text{a. Volumetric eff.} = \frac{Q_a}{Q_i} \times 100$$

$$\begin{aligned} \text{Ideal flow rate } (Q_i) &= \text{Nominal Displ.} \times \text{Speed} \\ &= \frac{10^{-5} \text{ m}^3}{\text{rev}} \times \frac{25 \text{ rev}}{\text{sec}} \\ &= 25 \times 10^{-5} \text{ m}^3 \text{ sec}^{-1} \end{aligned}$$

$$\begin{aligned} \text{Vol. eff.} &= \frac{Q_a}{Q_i} \times 100 \\ &= \frac{1/6000}{25 \times 10^{-5}} \times 100 \\ &= 66.67\% \end{aligned}$$

$$\text{b. Fluid Power} = P_a \times \Delta P$$

$$= \frac{1}{6000} \times 12 \times 10^5 = 200 \text{ kW}$$

$$\text{c. Shaft Power} = T \cdot \omega$$

$$T = \text{torque} \quad \omega = \text{angular speed}$$

$$\omega = 2\pi \nu$$

$$= 2\pi \times 25 = 500\pi \text{ rad/sec}$$

$$\text{S.P} = 12.5 \times 500\pi$$

$$= 625\pi = 1963.495 \text{ kW}$$

$$\text{d. Overall eff.} = \frac{FP}{SP} \times 100$$

$$= \frac{200}{625\pi} \times 100$$

$$= 10.1859\% \approx 10\%$$

2. A pump delivers  $35 \text{ dm}^3/\text{min}$  with a pressure change of  $100 \text{ bar}$ . If the overall efficiency is  $87\%$ , Calc. the shaft power

Soln

$$Q = 35 \text{ dm}^3/\text{min}$$

$$= \frac{35 \text{ dm}^3}{\text{min}} \times \frac{\text{min}}{60 \text{ sec}} \times \frac{\text{m}^3}{10^3 \text{ dm}^3}$$

$$= \frac{7}{12000} \cdot \text{m}^3 \text{ sec}^{-1}$$

$$\text{SP} = 100 \text{ bar} = 100 \times 10^5 \text{ Nm}^{-2}$$

$$= 10^7 \text{ Nm}^{-2}$$

$$\text{Over. effie} = 87\%$$

$$\text{Fluid Power} = Q_a \times \text{SP}$$

$$= \frac{7}{12000} \times 10^7$$

$$= \frac{17500}{3} \text{ kJ} = 5833.3 \text{ kJ}$$

$$\text{O.E} = \frac{\text{FP}}{\text{SP}} \times 100$$

$$\text{SP} = \frac{\text{FP} \times 100}{\text{O.E}}$$

$$= \frac{17500 \times 100}{87}$$

$$\text{SP} = 6704.98 \text{ kJ}$$

3. A pump has a nominal displacement of  $50 \text{ cm}^3/\text{rev}$  and a pressure rise of  $100 \text{ bar}$ . If the shaft power is  $15 \text{ kW}$ , Calculate the overall efficiency and volumetric efficiency. Taking the actual flow rate =  $35 \text{ dm}^3/\text{min}$  and speed of rotation =  $850 \text{ rpm}$ .

Soln

$$N.D = 50 \text{ cm}^3/\text{rev}$$

$$\frac{50 \text{ cm}^3}{\text{rev}} \times \frac{\text{min}}{10^6 \text{ cm}^3}$$

$$= 50 \times 10^6 \text{ m}^3 \text{ rev}^{-1}$$

$$\text{SP} = 100 \text{ bar} = 10^7 \text{ Nm}^{-2}$$

$$\text{S.P} = 15 \text{ kW} = 15 \times 10^3 \text{ kJ}$$

$$Q_a = \frac{35 \text{ dm}^3}{\text{min}} \times \frac{\text{min}}{10^3 \text{ dm}^3} \times \frac{\text{min}}{60 \text{ sec}}$$

$$= \frac{7}{12000} \text{ m}^3/\text{sec}$$

$$V(\text{speed}) = \frac{850 \text{ rev}}{\text{min}} \times \frac{\text{min}}{60 \text{ sec}}$$

$$= 85/6 \text{ rev/sec}$$

$$\text{Ideal } Q (Q_i) = N.D \times V$$

$$= 50 \times 10^6 \times 85/6$$

$$= 17/24000 \cdot \text{m}^3 \text{ sec}^{-1}$$

$$\text{Volumetric efficiency} = \frac{Q_{\text{actual}} \times 100}{Q_{\text{ideal}}}$$

$$= \frac{7/12000}{17/24000}$$

$$=$$

Fluid Power

Overall effie

4. Water is in which 2,40000cm at the rate outlet of the level and produce a 10 drive a type of the 40 66m/sec a. Power of b. Power c. Head use d. Efficiency nozzle m

$$= \frac{1/12000}{17/24000} \times 100$$

$$= \frac{1400}{17} \approx 82.35\%$$

$$z = 24,000 \text{ cm}$$

$$24,000 \text{ cm} \times \frac{1 \text{ m}}{100 \text{ cm}}$$

$$= 240 \text{ m}$$

Fluid Power =  $Q_a \times \rho P$

$$= 1/12000 \times 10^7$$

$$= \frac{17500}{3} = 5833.33 \text{ kJ}$$

$$Q = 13 \text{ litres/sec}$$

$$\frac{13 \text{ litres}}{\text{sec}} \times \frac{1 \text{ m}^3}{1000 \text{ litres}}$$

$$= 13 \times 10^{-3} \text{ m}^3/\text{sec}$$

Overall efficiency =  $\frac{F.P}{S.P} \times 100$

$$= \frac{17500/3}{15 \times 10^3} \times 100$$

$$= \frac{350}{9} \approx 38.89\%$$

Velocity of jet  $V_j = 66 \text{ ms}^{-1}$

Since the jet issuing from the nozzle will be at atmospheric pressure and datum level, hence.

$$P = 0, z = 0$$

$$\rho_{\text{water}} = 1000 \text{ kg/m}^3$$

Water is drawn from a reservoir in which the water level is 24,000 cm above the datum line at the rate of 13 litres/sec. The outlet of the pipe is at datum level and is fitted a nozzle to produce a high speed jet in order to drive a turbine of pelton wheel type. If the velocity of the jet is 66 m/sec. Calculate

$$a. P = \rho g Q \left( \frac{P}{\rho g} + \frac{v^2}{2g} + z \right)$$

as  $P = 0$  and  $z = 0$ .

$$P = \frac{\rho g Q v^2}{2g} = \frac{\rho Q v^2}{2}$$

$$= \frac{1000 \times 13 \times 10^{-3} \times 66^2}{2}$$

$$= 28314 \text{ kJ}$$

a. Power of jet

b. Power supplied from reservoir

c. Head used to overcome losses

d. Efficiency of the pipeline and nozzle in transmitting operation.

b. As the reservoir operates at atmospheric pressure. so.

$$P = 0, v = 0, z = 240 \text{ m}$$

$$P = \rho g Q z$$

$$= 1000 \times 9.81 \times 13 \times 10^{-3} \times 240$$

$$= 30607.2 \text{ kJ}$$

a. Power loss in transmission =  
 Power of reservoir - Power of jet  
 $= (30607.2 - 28314) \text{ kJ}$   
 $= 2293.2 \text{ kJ}$

Recall that

$$P = \rho g Q H; \quad H = \frac{P}{\rho g Q}$$

$$\text{Head loss } S = \frac{\text{Power loss}}{\rho g Q}$$

$$= \frac{2293.2}{1000 \times 9.81 \times 3 \times 10^{-3}}$$

$$= 17.982 \text{ m}$$

3. Oil of specific gravity 0.89 is drawn from a reservoir in which the oil is 30000m above the datum at the rate of 220 litres/sec. If the velocity of jet is 7m/sec, calculate

- Power of the jet
- Power applied from the reservoir
- Head used to overcome losses
- Efficiency of the pipeline and nozzle in transmitting operation.

**ATTENTION! ATTENTION! ATTENTION!**

No 4d. → Please below is 4d.

$$\text{Efficiency} = \frac{\text{Power of jet}}{\text{Power of reservoir}} \times 100$$

$$= \frac{28314}{30607.2} \times 100$$

$$= 92.507\%$$

Soln To No 5

a. Jet is at atmospheric pressure. So  
 $P = 0, \quad v = 7 \text{ m/s}, \quad z = 0.$

$$P = \rho g Q v^2 = \frac{\rho g Q v^2}{2}$$

But recall that

$$Q = 220 \times 10^{-3} \text{ m}^3/\text{sec}$$

$$S.G. = \frac{\rho_{oil}}{\rho_{water}}$$

$$\rho_{oil} = S.G. \times \rho_{water}$$

$$= 0.89 \times 1000 = 890 \text{ kg/m}^3$$

$$P = \frac{890 \times 220 \times 10^{-3} \times 7^2}{2}$$

$$= 4797.11 \text{ kJ}$$

b. Power from reservoir

$$P = 0, \quad v = 0, \quad z = 30000 \text{ m (30000)}$$

$$P = \rho g Q z$$

$$= 890 \times 9.81 \times 220 \times 10^{-3} \times 30000$$

$$= 576239.4 \text{ kJ}$$

c. Loss in power head

$$= (576239.4 - 4797.11) \text{ kJ}$$

$$= 571442.3 \text{ kJ}$$

$$h = \frac{P}{\rho g Q} = \frac{571442.3}{890 \times 9.81 \times 220 \times 10^{-3}}$$

$$= 297.50 \text{ m}$$

d. Efficiency =

$$= \frac{4797.11}{576239.4}$$

6. A water jet of diameter 100mm is drawn from the base of a dam of height 20m. Calculate the velocity of the jet at the base of the dam.

Soln

$$h = 20 \text{ m}$$

$$d = 100 \text{ mm}$$

$$\rho = 1000 \text{ kg/m}^3$$

$$A = \frac{\pi d^2}{4}$$

$$v_f = 0$$

$$v_f^2 = v_i^2$$

$$v_i = v_f$$

$$Q = VA$$

$$= uA$$

$$h = \frac{v^2}{2g}$$

$$4. \text{ Efficiency} = \frac{\text{Power of jet}}{\text{Power of } P}$$

$$= \frac{4797.1}{516239.4} \times 100$$

$$= 0.93\%$$

$$= 1000 \times 9.81 \times 20 \times 0.1556$$

$$= 30529.8 \text{ W}$$

phenz

= 0.

$$\frac{Qv^2}{2}$$

6. A water fountain sends a stream of water 20m up into the air. At the base of the stream is 10cm in diameter, what power is required to send the water to this height?

Soln

$$h = 20 \text{ m}$$

$$d = 10 \text{ cm} = 0.10 \text{ m}$$

$$\rho = 1000 \text{ kg m}^{-3}$$

$$A = \frac{\pi d^2}{4} = \frac{\pi \times 0.10^2}{4} = \frac{\pi}{400}$$

$$\approx 7.85 \times 10^{-3} \text{ m}^2$$

$v_f = 0$  (momentarily at rest at a max. height)

$$v_f^2 = v_i^2 - 2gh$$

$$v_i = \sqrt{v_f^2 + 2gh}$$

$$\sqrt{0^2 + 2 \times 9.81 \times 20}$$

$$= 19.80 \text{ ms}^{-1}$$

$$= 19.80 \text{ ms}^{-1}$$

$$Q = VA$$

$$= \frac{\pi}{400} \times 19.80$$

$$= 0.1556 \text{ m}^3 \text{ sec}^{-1}$$

$$W = \rho g Q z$$

7. A venturimeter with an entrance diameter of 0.3m and throat diameter of 0.2m is used to measure the volume of gas flowing through a pipe. The discharge coefficient is 0.96. Assuming the specific weight of the gas to be constant at  $19.62 \text{ N/m}^3$ , calculate the volume flowing when the pressure difference between entrance and the throat is measured as 0.06m on a water U-tube manometer.

Soln

$$D_1 = 0.3 \text{ m}$$

$$D_2 = 0.2 \text{ m}$$

$$C_d = 0.96$$

$$\omega_g = 19.62 \text{ N/m}^3$$

$$Q = ?$$

$$Q = C_d A_1 A_2 \sqrt{\frac{2gh}{\sqrt{A_1^2 - A_2^2}}}$$

$$h_g = \frac{h_{w0}}{h_g} = \frac{\omega_w}{\omega_w}$$

$$h_g = \frac{\omega_w}{\omega_g} \times h_{w0}$$

$$= \frac{9.81 \times 1000 \text{ N/m}^3 \times 0.06}{19.62 \text{ N/m}^3}$$

$$= 30 \text{ m of gas}$$

$$Q = \frac{C_d \times \frac{\pi d_1^2}{4} \times \frac{\pi d_2^2}{4} \times \sqrt{2gh}}{\sqrt{\left(\frac{\pi d_1^2}{4}\right)^2 - \left(\frac{\pi d_2^2}{4}\right)^2}}$$

$$= \frac{0.96 \times \frac{\pi^2}{16} \times 0.3^2 \times 0.2 \times \sqrt{2 \times 9.81 \times 30}}{\sqrt{\left(\frac{\pi \times 0.3^2}{4}\right)^2 - \left(\frac{\pi \times 0.2^2}{4}\right)^2}}$$

$$= 0.816 \text{ m}^3/\text{s}$$

$$= 0.816 \text{ m}^3/\text{s}$$

8. A venturimeter of throat diameter 0.076m is fitted into a 0.125m diameter vertical pipe in which liquid of relative density 0.8 flows downwards. Pressure gauges are fitted to the inlet and to the throat section, the throat 0.914 below the inlet. Taking the coefficient of the meter as 0.97. Find the discharge

a. when the pressure gauge read the same.

b. when the inlet gauge reads  $15170 \text{ N/m}^2$  higher than the throat gauge.

$$d_1 = 0.125 \text{ m}$$

$$d_2 = 0.076 \text{ m}$$

$$z_1 = 0.914 \text{ m}$$

$$z_2 = 0 \text{ m}$$

$$C_d = 0.97$$

$$Q = \frac{C_d A_1 A_2 \sqrt{2gh}}{\sqrt{A_1^2 - A_2^2}}$$

For a vertical venturimeter

$$h = \left\{ \frac{P_1 - P_2}{\rho g} + (z_1 - z_2) \right\}$$

a. Pressure gauges read the same hence  $P_1 = P_2$ .

$$h = \left\{ \frac{P_1 - P_2}{\rho g} + (z_1 - z_2) \right\}$$

$$R.D = \frac{\text{Specific weight of liquid}}{\text{Specific weight of water}}$$

$$0.8 = \frac{\rho g}{9.81 \times 1000 \text{ N/m}^2}$$

$$\rho g = 0.8 \times 9.81 \times 1000 \text{ N/m}^2 = 7848 \text{ N/m}^2$$

$$h = \left\{ \frac{0}{7848} + (0.914 - 0) \right\}$$

$$= 0.914 \text{ m}$$

$$Q = \frac{C_d \frac{\pi d_1^2}{4} \times \frac{\pi d_2^2}{4} \times \sqrt{2gh}}{\sqrt{\left(\frac{\pi d_1^2}{4}\right)^2 - \left(\frac{\pi d_2^2}{4}\right)^2}}$$

$$= \frac{C_d \frac{\pi^2}{16} d_1^2 d_2^2 \sqrt{2gh}}{\sqrt{\left(\frac{\pi d_1^2}{4}\right)^2 - \left(\frac{\pi d_2^2}{4}\right)^2}}$$

$$= \frac{C_d \frac{\pi^2}{16} d_1^2 d_2^2 \sqrt{2gh}}{\sqrt{\left(\frac{\pi d_1^2}{4}\right)^2 - \left(\frac{\pi d_2^2}{4}\right)^2}}$$

$$= \frac{C_d \frac{\pi^2}{16} d_1^2 d_2^2 \sqrt{2gh}}{\sqrt{\left(\frac{\pi d_1^2}{4}\right)^2 - \left(\frac{\pi d_2^2}{4}\right)^2}}$$

b. Pressure of outlet  $C_d$

$$h = \frac{P_1 - P_2}{\rho g}$$

$$= \frac{15170}{7848}$$

$$= 1.92$$

$$Q = C_d \frac{\pi d_1^2}{4} \times \frac{\pi d_2^2}{4} \times \sqrt{2gh}$$

$$= 0.97 \times \frac{\pi^2}{16} \times 0.125^2 \times 0.076^2 \times \sqrt{2 \times 7848 \times 1.92}$$

$$= \frac{\pi^2 \times 0.125^2 \times 0.076^2 \times \sqrt{2 \times 7848 \times 1.92}}{16}$$

$$= 0.019$$

B. Pressure outlet  $P_1 - P_2$

$$h = \frac{P_1 - P_2}{\rho g}$$

$$= \frac{15170}{7848}$$

$$= 1.92$$

$$Q = C_d \frac{\pi d_1^2}{4} \times \frac{\pi d_2^2}{4} \times \sqrt{2gh}$$

b. Pressure difference inlet and outlet  $C_d = 15170 \text{ Nm}^{-2}$   $P_1 - P_2 = 15170$

$$= \frac{0.97 \times \bar{u}^2 \times 0.152^2 \times 0.076^2 \times \sqrt{2 \times 9.81 \times 2.847}}{16}$$

$$h = \frac{P_1 - P_2}{\rho} + z_1 - z_2$$

$$= \frac{15170}{7848} + (0.914 - 0)$$

$$= 2.847 \text{ m}$$

$$Q = C_d \frac{\bar{u}^2 d_1^2 \times \bar{u} d_2^2 \times \sqrt{2gh}}{4}$$

$$= \frac{0.97 \times \bar{u}^2 \times 0.152^2 \times 0.076^2 \times \sqrt{2 \times 9.81 \times 2.847}}{16}$$

$$\sqrt{\left(\frac{\bar{u} \times 0.152^2}{4}\right)^2 - \left(\frac{\bar{u} \times 0.076^2}{4}\right)^2}$$

$$= 0.0192 \text{ m}^3/\text{s}$$

B. Pressure difference, inlet and outlet  $P_1 - P_2 = 15170 \text{ Nm}^{-2}$

$$h = \frac{P_1 - P_2}{\rho} + (z_1 - z_2)$$

$$= \frac{15170}{7848} + (0.914 - 0)$$

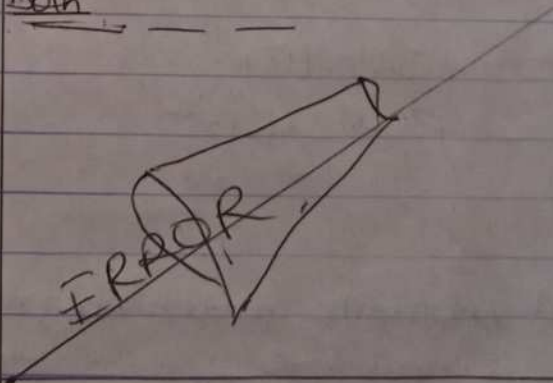
$$= 2.847 \text{ m}$$

$$Q = C_d \frac{\bar{u}^2 d_1^2 d_2^2 \sqrt{2gh}}{16}$$

$$\sqrt{\left(\frac{\bar{u} d_1^2}{4}\right)^2 - \left(\frac{\bar{u} d_2^2}{4}\right)^2}$$

9. The water is flowing through a tapering pipe having diameter 300mm and 150mm at section 1 and 2 respectively. The discharge through the pipe is 40 litre/sec. The section 1 is 10m above the datum and section 2 is 6m above the datum. Find the intensity of the pressure at section 2 if that at section 1 is 400 kN/m<sup>2</sup>.

Soln



$$Q = \frac{40 \text{ litre}}{\text{sec}} \times \frac{\text{m}^3}{1000 \text{ litre}}$$

$$= 0.04 \text{ m}^3/\text{sec}$$

$$z_1 = 10 \text{ m} \quad z_2 = 6 \text{ m}$$

$$P_1 = 400 \text{ kN/m}^2 \quad P_2 = ?$$

Recall that

$$Q = A_1 v_1 = A_2 v_2$$

$$v_1 = \frac{Q}{A_1} \quad ; \quad v_2 = \frac{Q}{A_2}$$

$$= \frac{0.04}{4 (300 \times 10^{-3})^2} \quad ; \quad \frac{0.04}{4 (150 \times 10^{-3})^2}$$

$$v_1 = 0.566 \text{ m/s} \quad ; \quad v_2 = 2.26 \text{ m/s}$$

According to Bernoulli's eqn.

$$\frac{P_1}{\rho} + \frac{v_1^2}{2g} + z_1 = \frac{P_2}{\rho} + \frac{v_2^2}{2g} + z_2$$

$$P_2 = \left\{ \frac{P_1}{\rho} + (z_1 - z_2) + \frac{(v_1^2 - v_2^2)}{2g} \right\} \rho$$

$$= \left\{ \frac{400 \text{ kN/m}^2}{9.81} + (10 - 6) + \frac{(0.566^2 - 2.26^2)}{2(9.81)} \right\} \times 9.81 \text{ kN/m}^3$$

$$P_2 = 44.52 \times 9.81$$

$$= 436.83 \text{ kN/m}^2$$

10. A submarine moves horizontally in sea, and has a axis 15m below the surface of water. A pitot tube properly placed just in front of the submarine and

along its axis is connected to the 2 limbs of U-tube containing mercury. The difference in mercury is found to be 170mm. Find the speed of the submarine knowing that the specific gravity of mercury is 13.6 and that of seawater is 1.026 with respect to that of freshwater.

$$v = \sqrt{2g\Delta h}$$

where

$v$  = velocity  $\Delta h$  = diff. in pressure head

Read

$y$  = manometric reading

$$\Delta H_{Hg} = 13.6$$

$$\rho_{sw} = 1.026$$

$$y = 170 \text{ mm} = 0.17 \text{ m}$$

$$v = y \left( \frac{\Delta H_{Hg}}{\rho_{sw}} - 1 \right)$$

$$= 0.17 \left( \frac{13.6}{1.026} - 1 \right)$$

$$= 2.08 \text{ m}$$

$$v = \sqrt{2 \times 9.81 \times 2.08}$$

$$= 6.39 \text{ m/s}$$