

EKOK NZIE OKPOKAM
18/ENG05/045
FLUID MECHANICS

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MECHATRONICS

- i] Speed of rotation = 1500 rev/min
nominal displacement = 10 cm³/rev
Torque input = 12.5 Nm
Pressure change = 12 bar
Rate = 10 dm³/min

Solution

$$\text{Ideal flow rate} = \text{Nominal displacement} \times \text{Speed}$$

$$\begin{aligned} \text{Ideal flow rate} &= \text{Nominal displacement} \times \text{Speed} \\ 10 \times 1500 &= 15000 \text{ cm}^3/\text{min} \\ &= 15 \text{ dm}^3/\text{min} \end{aligned}$$

$$\text{Volumetric efficiency} = \frac{\text{Actual flow}}{\text{Ideal flow}}$$

$$= \frac{10}{15} = 0.67 \text{ or } 67\%$$

$$Q = (10 \times 10^{-3}) / 60 \text{ m}^3/\text{s} = 16.7 \times 10^{-6} \text{ m}^3/\text{s}$$

$$\Delta p = 12 \times 10^5 \text{ N/m}^2$$

$$\text{ii] Fluid power} = Q \Delta p$$

$$16.7 \times 10^{-6} \times 12 \times 10^5 = 20.04 \text{ watts}$$

$$\text{iii] Shaft power} = 2\pi NT/60$$

$$2\pi \times 1500 \times 12.5 / 60$$

$$= 1963.5 \text{ Nm}$$

$$\text{iv] Overall efficiency} = \frac{F.P}{S.P}$$

$$= \frac{20.04}{1963.5}$$

$$= 0.01$$

- 2] delivers 35 dm³/min
pressure change = 100 bar
efficiency = 87%
Shaft power = 2πNT/60
F.P = Q ΔP

$$Q = (35 \times 10^{-3}) / 60 = 5.8 \times 10^{-4}$$

$$\Delta P = 100 \times 10^5 \text{ N/m}^2$$

$$F.P = 5800 \text{ watts}$$

$$\text{Efficiency} = \frac{F.P}{S.P}$$

$$0.87 = \frac{5800}{S.P}$$

$$\therefore S.P = \frac{5800}{0.87} = 6590 \text{ Nm}$$

- 3] Nominal displacement = 50 cm³/rev

$$\text{pressure} = 100 \text{ bar}$$

$$\text{Shaft power} = 15 \text{ kilowatts}$$

$$\text{Actual flow rate} = 35 \text{ dm}^3/\text{min}$$

$$\text{Speed} = 850 \text{ r.p.m}$$

Solution

$$\text{Ideal flow rate} = 50 \times 850 = 42.5 \text{ dm}^3/\text{min}$$

$$\text{Volumetric efficiency} = \frac{35}{42.5} = 0.82$$

$$\text{Fluid power} = \left(\frac{35 \times 10^{-3}}{60} \right) \times 100 \times 10^5$$

$$= 5.8 \times 10^{-10}$$

$$\text{Overall efficiency} = \frac{5.8 \times 10^{-10}}{15 \times 10^{-3}}$$

$$= 3.9 \times 10^{-8}$$

4] $S_g =$
 $\rho = 1000 \times 1000 = 1000 \text{ kg/m}^3$
 $Z = 240 \text{ m}$
 $Q = 13 \text{ ltr/sec} = 13 \times 10^{-3} \text{ m}^3/\text{sec}$
 Velocity of jet = 0.6 m/s
 Since outlet of pipe is at datum
 $p = 0$ $Z = 0$

$$P = \rho Q + \frac{\rho Q v^2}{2} + \rho g Q Z$$

$$P = 1000 + 13 \times 10^{-3} \times (0.6)^2$$

$$P = 28314 \text{ watts}$$

ii) Power supplied through reservoir
 Substituting $p = 0$ and $Z = 0$
 in eq 1
 $P = \rho g Q Z = 1000 \times 9.81 \times 13 \times 10^{-3} \times 240$
 $= 30607.2 \text{ watts}$

iii) Head loss, $h = \frac{\text{Power lost in transmission}}{\rho g Q}$
 $= \frac{(30607.2 - 28314)}{1000 \times 9.81 \times 13 \times 10^{-3}}$
 $h = 17.982 \text{ m}$

iv) Efficiency = $\frac{\text{Power of jet}}{\text{Power of reservoir}} \times 100$

$$= \frac{28314}{30607.2} \times 100$$

$$= 92.5\%$$

v) Eff

5] $S_g = 0.89$
 $\rho = 890$
 $Z = 300 \text{ m}$
 $Q = 0.22 \text{ m}^3/\text{sec}$
 Velocity = 7 m/s
 $P = \rho Q + \frac{\rho Q v^2}{2} + \rho g Q Z$

(i) Power of jet =
 $P = \frac{\rho Q v^2}{2} = \frac{890 \times 0.22 \times (7)^2}{2}$
 $= 4797.1 \text{ watts}$

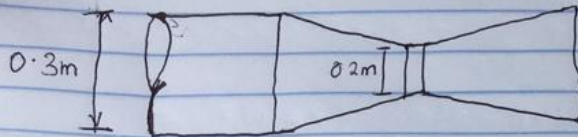
(ii) Power supplied from reservoir
 $P = \rho g Q Z$
 $= 890 \times 9.81 \times 0.22 \times 300$
 $= 576239.4 \text{ watts}$

(iii) Head loss, $h = \frac{\text{Power lost in transmission}}{\rho g Q}$

$$= \frac{576239.4 - 4797.1}{0.22 \times 890 \times 9.81}$$

$$= 29.75 \text{ m}$$

v) Efficiency = $\frac{\text{Power of jet}}{\text{Power of reservoir}} \times 100$
 $= \frac{4797.1}{576239.4} \times 100 = 0.83$



$$\text{Diameter of inlet} = 0.3\text{m}$$

$$\text{Area of inlet} = \frac{\pi \times 0.3^2}{4} = 0.071\text{m}^2$$

$$y = 0.06\text{m}$$

$$\text{Diameter of throat} = 0.2\text{m}$$

$$\text{Area of throat} = \frac{\pi \times 0.2^2}{4} = 0.031\text{m}^2$$

$$\text{The differential reading (h)} = y$$

$$\text{Specific weight of gas} = 19.62\text{N/m}^3 = 0.01962\text{kN/m}^3$$

$$\therefore \text{Specific gravity of gas} = \frac{0.01962}{9.81} = 2 \times 10^{-3}$$

$$\text{The differential reading (h)} = \frac{y}{2} \left[\frac{\text{Sp. W}}{\text{Sp. G}} - 1 \right]$$

$$= 0.06 \left[\frac{1}{2 \times 10^{-3}} - 1 \right]$$

$$= 29.94$$

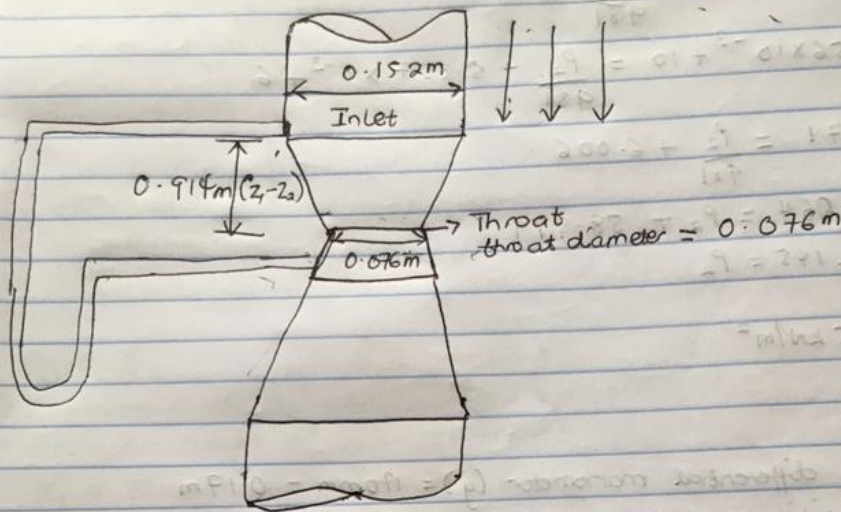
$$Q = C_d \times \frac{A_1 A_2}{\sqrt{A_1^2 - A_2^2}} \times \sqrt{2gh}$$

$$= 0.96 \times \frac{0.071 \times 0.031}{\sqrt{0.071^2 - 0.031^2}} \times \sqrt{2 \times 9.81 \times 29.94}$$

$$= 0.96 \times 0.0345 \times 24.24 =$$

$$= 0.803\text{m}^3/\text{s}$$

Number 8



$$\text{Diameter of inlet} = 0.152 \text{ m}$$

$$\text{Area of inlet} = \frac{\pi \times 0.152^2}{4} = 0.018 \text{ m}^2$$

$$\text{Diameter of throat} = 0.076 \text{ m}$$

$$\text{Area of throat} = \frac{\pi \times 0.076^2}{4} = 4.54 \times 10^{-3} \text{ m}^2$$

$$\text{The differential reading (h)} = \left[\frac{P_1}{\rho} + z_1 \right] - \left[\frac{P_2}{\rho} + z_2 \right]$$

$$h = \left[\frac{P_1}{\rho} - \frac{P_2}{\rho} \right] - [z_1 - z_2]$$

a) when the pressure gauge reading is the same

$$\text{i.e. } \left[\frac{P_1}{\rho} - \frac{P_2}{\rho} \right] = 0$$

$$\therefore h = 0 - [z_1 - z_2] \quad \therefore h = [z_1 - z_2]$$

$$= 0 - [0.194]$$

$$= -0.194 \text{ m}$$

Number 8 Contd.

$$\therefore \text{discharge } Q = C_d \times \frac{A_1 \times A_2}{\sqrt{A_1^2 - A_2^2}} \times \sqrt{2gh}$$

$$= 0.97 \times 0.018 \times 4.54 \times 10^{-3} \times \sqrt{2 \times 9.81 \times 0.194}$$

$$\sqrt{0.018^2 - (4.54 \times 10^{-3})^2}$$

$$= 0.97 \times 4.692 \times 10^{-3} \times 1.951$$

$$= 8.88 \times 10^{-3} \text{ m}^3/\text{s}$$

b) when the inlet gauge reads 15170 N/m^2 higher than the throat gauge

i.e

$$\left[\frac{P_1}{\rho} - \frac{P_2}{\rho} \right] = 15170 \text{ N/m}^2 \quad \therefore P_1 - P_2 = 15170 \text{ N/m}^2$$

$$\Rightarrow \frac{P_1 - P_2}{\rho} = \frac{15170}{\rho} = \frac{15170}{9.81} =$$

$$\Rightarrow \frac{P_1 - P_2}{\rho} = 1546.38 \text{ m}$$

$$\therefore h = \frac{P_1 - P_2}{\rho} - [z_1 - z_2]$$

$$\therefore h = 1546.38 - 0.194$$

$$= 1546.2 \text{ m}$$

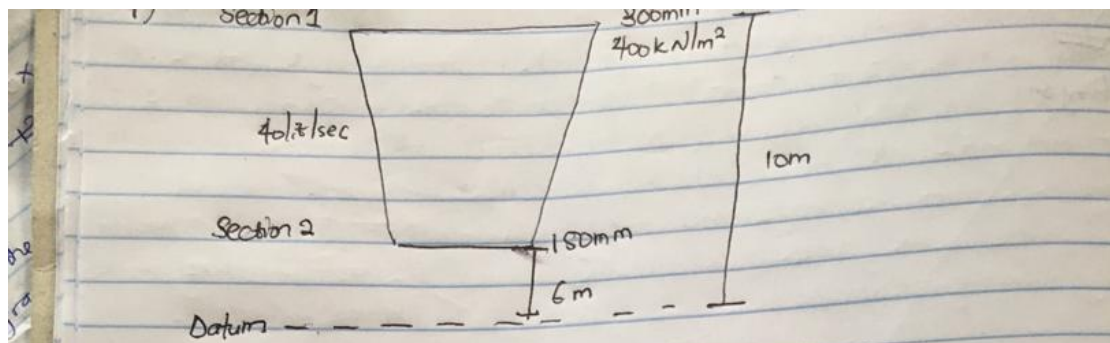
$$\therefore Q = C_d \times \frac{A_1 \times A_2}{\sqrt{A_1^2 - A_2^2}} \times \sqrt{2gh}$$

$$Q = 0.97 \times 0.018 \times 4.54 \times 10^{-3} \times \sqrt{2 \times 9.81 \times 1546.2}$$

$$\sqrt{0.018^2 - (4.54 \times 10^{-3})^2}$$

$$= 0.798 \text{ m}^3/\text{s}$$

$$\frac{m}{(P_1 - P_2)} \times \rho = \frac{P_1 - P_2}{\rho}$$



Section 1	Section 2
(d ₁) Diameter = 300 mm	(d ₂) Diameter = 150 mm
(P ₁) Pressure = 400 kN/m ²	(P ₂) Pressure = ?
Z ₁ = 10 m	Z ₂ = 6 m

Volume rate (Q) = 40 lit/sec = 0.04 m³/s

$$Q = 0.04 \text{ m}^3/\text{s}$$

Recall $Q = A_1 V_1 = A_2 V_2$

$$\text{But } A_1 = \frac{\pi d_1^2}{4} = \frac{\pi \times 0.3^2}{4} = 0.236 \text{ m}^2$$

$$\therefore 0.04 = 0.236 \times V_1$$

$$V_1 = 0.169 \text{ m/s}$$

$$\text{But } A_2 = \frac{\pi d_2^2}{4} = \frac{\pi \times 0.15^2}{4} = 0.118 \text{ m}^2$$

$$V_2 = \frac{0.04}{0.118} = 0.339 \text{ m/s}$$

$$0.118$$

From Bernoulli's equation

$$\frac{P_1}{\rho} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\rho} + \frac{V_2^2}{2g} + Z_2$$

$$\frac{400}{9.81} + \frac{0.169^2}{2 \times 9.81} + 10 = \frac{P_2}{9.81} + \frac{0.339^2}{2 \times 9.81} + 6$$

$$\cancel{40.77} + \cancel{1.456} + 10 = \frac{P_2}{9.81} +$$

$$40.77 + 1.456 \times 10^{-3} + 10 = \frac{P_2}{9.81} + 5.857 \times 10^{-3} + 6$$

$$50.771 = \frac{P_2}{9.81} + 6.006$$

$$498.064 = P_2 + 58.919$$

$$439.145 = P_2$$

$$P_2 = 439.145 \text{ kN/m}^2$$

10) Reading of differential manometer (y) = 170 mm = 0.17 m

~~Reading~~ Specific gravity of mercury = 13.6

Specific gravity of ^{sea}water = 1.025

$$\text{to find head (h)} = \frac{0.17}{2} \left[\frac{13.6}{1.025} - 1 \right] = 2.09$$

$$= \cancel{0.17} \times 2.0$$

$$\therefore \text{Velocity of submay} = \sqrt{2gh} = \sqrt{2 \times 9.81 \times 2.09} = 6.403 \text{ m/s}$$