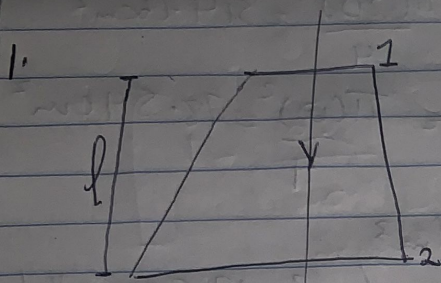


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13/ENG03/047

CIVIL ENGINEERING



length = 2.0m - l

Velocity flow at smaller end = 5m/s - v_1

velocity flow at lower end = 2m/s - v_2

Let the pressure head at the smaller end

$$= P_s$$

$P_s = 2.5m$ of liquid

Let Loss of the head = $h_L = 0.35(v_1 - v_2)^2$

$$= \frac{0.35(5-2)^2}{2 \times 9.81} = 0.161m$$

Pressure at lower end, $P_L = ?$

Applying Bernoulli's Equation

$$\frac{P_1}{\rho g} + \frac{v_1^2}{2g} + z_1 = \frac{P_2}{\rho g} + \frac{v_2^2}{2g} + z_2 + h_L$$

Where $P_s = \frac{P_1}{\rho g}$ and $P_L = \frac{P_2}{\rho g}$

$z_1 = 2.0$ and $z_2 = 0$

Substituting in values

$$2.5 + \frac{5^2}{2 \times 9.81} + 2.0 = P_L + \frac{2^2}{2 \times 9.81} + 0 + 0.161$$

$$2.5 + \frac{25}{19.62} + 2 = PL + \frac{4}{19.62} + 0.161$$

$$2.5 + \frac{25}{19.62} + 2 = PL + \frac{4}{19.62} + 0.161$$

$$2.5 + \frac{25}{19.62} + 2 - \left(\frac{4}{19.62} + 0.161 \right) = PL$$

$$5.74 - 0.365 = PL$$

$$PL = 5.409 \text{ m}$$

2. Let inlet diameter $= D_1 = 20 \text{ cm}$

Let throat diameter $= D_2 = 10 \text{ cm}$

Let inlet area $= A_1 = \frac{\pi D_1^2}{4} = \frac{\pi (20)^2}{4} = 314.16 \text{ cm}^2$

Let throat area $= A_2 = \frac{\pi D_2^2}{4} = \frac{\pi (10)^2}{4} = 78.54 \text{ cm}^2$

Density of water, $\rho = 1000 \text{ kg/m}^3$

Pressure at inlet $= 17.658 \text{ m/cm}^2 = 17.658 \times 10^4 \text{ N/m}^2$

$$\therefore \frac{P_1}{\rho g} = \frac{17.658 \times 10^4}{1000 \times 9.81} = 18 \text{ m}$$

$\frac{P_2}{\rho g} = -30 \text{ cm}$ of mercury, $S_{\text{mercury}} = 13.6$

$\frac{P_2}{\rho g} = -30 \times 10^{-2} \text{ m}$ of mercury $\times 13.6$

$$\frac{P_2}{\rho g} = -4.08$$

HD, differential head $= 18 - (-4.08)$

$$= 22.08 \sqrt{100}$$

$$= 22.08 \text{ cm}$$

using, $Q = \frac{C_d \sqrt{2gh} A_1 A_2}{\sqrt{A_1^2 - A_2^2}}$

$$= 0.98 \times 2081.37 \times 22674.1264$$

$$= 304.184112$$

$$= 165455.3 \text{ cm}^3/\text{s}$$

$$= 165.45 \text{ lit/sec}$$

3.

$$d_1 = 30 \text{ cm}$$

$$A_1 = \frac{\pi d_1^2}{4} = \frac{\pi (30)^2}{4} = 706.86 \text{ cm}^2$$

$$d_2 = 15 \text{ cm}$$

$$A_2 = \frac{\pi d_2^2}{4} = \frac{\pi (15)^2}{4} = 176.72 \text{ cm}^2$$

$$\rho_o = 0.9$$

$$S_{mg} = 13.6$$

manometer reading, $x = 50 \text{ cm}$ of mg

$$C_d = 0.64$$

$$\text{Differential level, } h = x \left(\frac{S_{mg}}{\rho_o} - 1 \right)$$

$$h = 50 \left(\frac{13.6}{0.9} - 1 \right)$$

$$h = 705.56 \text{ cm of oil}$$

The rate of flow of oil

$$Q = \frac{C_d \sqrt{2gh} \cdot A_1 A_2}{\sqrt{A_1^2 - A_2^2}}$$

$$Q = 0.64 \times \sqrt{2 \times 9.81 \times 705.56} \times 706.86 \times$$

$$\sqrt{(706.86)^2 - (176.72)^2}$$

$$Q = 137.44 \text{ lit/s}$$

4.

$$X = 170 \text{ mm} = 170 \times 10^{-3} = 0.17 \text{ m}$$

$$S_g = 13.6$$

$$S_o = 1.026$$

$V = ?$

$$h = X \left[\frac{S_g}{S_o} - 1 \right] = 0.17 \left[\frac{13.6}{1.026} - 1 \right]$$

$$= 2.0834 \text{ m}$$

$$V = \sqrt{2 \times 9.81 \times 2.0834} = 6.393 \text{ m/s}$$

$$\text{or } = 23.01 \text{ km/hr}$$

5.

$$Q = 0.05 \text{ m}^3/\text{min} = 50 \text{ dm}^3/\text{min}$$

$$P_o = 15 \text{ bar} = 15 \times 10^5 \text{ N/m}^2$$

$$V = 1700 \text{ rev/min}$$

$$T = 15 \text{ A/m} \quad \mu = 10 \text{ cm}^2/\text{rev}$$

(i) $V, T = \text{Actual flow rate}$

Ideal flow rate

$$T_{Fr} = \frac{10 \times 1700}{60000} = \frac{17000}{60000} = 0.017 \text{ m}^2/\text{min}$$

$$\therefore VE = \frac{0.05}{0.017} = 2.94\%$$

(ii) Fluid Power = $P \times Q$

$$P = 15 \times 10^5 \text{ N/m}^2$$

$$Q = 0.05 \text{ m}^3/\text{min} = \frac{0.05}{60}$$

$$\text{Fluid Power} = 15 \times 10^5 \times 8.33 \times 10^{-4} = 1.25 \times 10^2 \text{ W}$$

$$= 1249.5 \times 10^{-3}$$

$$= 1249.5 \text{ watts}$$

$$(ii) \text{ Shaft Power} = \frac{2\pi \cancel{NP}}{60} = \frac{2\pi}{60} \frac{2\pi NP}{60}$$

$$= \frac{2\pi \times 1700 \times 15}{60}$$

$$= 2670.35 \text{ watts}$$

$$\text{Overall efficiency} = \frac{\text{Fluid power}}{\text{Shaft power}}$$

$$= \frac{1249.5}{2670.35}$$

$$= 0.468\%$$