

LAWO DANILOLA

MECHATRONICS

18/ENG05/058

FLUID MECHANICS ASSIGNMENT.

- 1) Rate of Pump delivery = $10 \text{ dm}^3/\text{min}$
Pressure change = $12 \text{ bar} = 12 \times 10^5 \text{ N/m}^2$
Speed of rotation = $1500 \text{ rev/min} = \frac{1500}{60 \text{ sec}} = 25 \text{ rev/sec}$
Nominal displacement = $10 \text{ cm}^3/\text{rev}$
Torque input = 12.5 Nm

i) Volumetric efficiency = $\frac{\text{Actual flow rate}}{\text{Ideal flow rate}} \times 100\%$

$$\text{Actual flow rate} = 10 \text{ dm}^3/\text{min} = \frac{10 \text{ dm}^3}{\text{min}} \times \frac{1 \text{ min}}{60 \text{ sec}} = 1.67 \times 10^{-4} \text{ m}^3/\text{sec}$$

$$\text{Nominal displacement} = \frac{10 \text{ cm}^3}{1 \text{ rev}} \times \frac{1 \text{ m}^3}{1000 \text{ cm}^3} = 1 \times 10^{-5} \text{ m}^3/\text{rev}$$

$$\text{Ideal flow rate} = \text{nominal displacement} \times \text{speed} = 1 \times 10^{-5} \times 25 = 2.5 \times 10^{-4} \text{ m}^3/\text{sec}$$

$$\text{Volumetric efficiency} = \frac{1.67 \times 10^{-4}}{2.5 \times 10^{-4}} \times 100 = 66.8\%$$

ii) Fluid power = Actual flow rate \times Pressure
 $= 1.67 \times 10^{-4} \times 12 \times 10^5 = 200 \text{ watts}$

iii) Shaft power = Torque input \times angular speed
Torque input = 12.5 Nm
Angular speed = $\omega = \frac{2\pi N}{60} = \frac{2\pi \times 1500}{60} \text{ rad/s}$

$$\text{Shaft power} = 12.5 \times 2 \times 3.142 \times 25 = 1963.5 \text{ watts}$$

$$\begin{aligned} \text{(iii) Overall efficiency} &= \frac{\text{Fluid power}}{\text{Shaft power}} \times 100\% \\ &= \frac{200}{1963.5} \times 100 \\ &= 10.19\% \end{aligned}$$

$$\begin{aligned} 2) \text{ Rate of delivery} &= 35 \text{ dm}^3/\text{min} = \frac{35}{60} = 0.583 \text{ dm}^3/\text{sec} \\ \text{Pressure change} &= 100 \text{ bar} = 100 \text{ bar} = 100 \times 10^5 \text{ N/m}^2 \\ \text{Overall efficiency} &= 87\% \\ \text{Shaft power} &= ? \end{aligned}$$

$$\text{Rate of delivery} = \frac{0.583}{1000} = 5.83 \times 10^{-4} \text{ m}^3/\text{sec}$$

$$\begin{aligned} \text{Fluid power} &= \text{Actual flow rate} \times \text{pressure change} \\ &= 5.83 \times 10^{-4} \times 100 \times 10^5 \\ &= 5833.33 \text{ watts} \end{aligned}$$

$$\text{Overall efficiency} = \frac{\text{Fluid power}}{\text{Shaft power}} \times 100$$

$$87 = \frac{5833.33}{\text{S.p}} \times 100$$

$$\begin{aligned} \frac{0.87}{5833.33} &= \frac{1}{\text{Shaft power}} \\ \therefore \text{Shaft power} &= 6704.977 \text{ watts} \end{aligned}$$

$$3) \text{ Nominal displacement} = 50 \text{ cm}^3/\text{rev} = 5 \times 10^{-4} \text{ m}^3/\text{rev}$$

$$\text{Pressure change} = 100 \text{ bar} = 100 \times 10^5 \text{ N/m}^2$$

$$\text{Shaft power} = 15 \text{ kW} = 15000 \text{ Watts}$$

$$\text{Actual flow rate} = 35 \text{ dm}^3/\text{min} = 5.833 \times 10^{-4} \text{ m}^3/\text{sec}$$

$$\text{Speed of rotation} = 850 \text{ rpm} = 14.2 \text{ rps}$$

$$\text{Overall efficiency} = \frac{\text{Fluid power}}{\text{Shaft power}} \times 100$$

$$\begin{aligned} \text{Fluid power} &= \text{Actual flow rate} \times \text{Pressure change} \\ &= 5.833 \times 10^{-4} \times 100 \times 10^5 \\ &= 5833 \text{ watts} \end{aligned}$$

$$\text{Shaft power} = \text{Torque input} \times \text{angular speed}$$

15000 Watts

$$\begin{aligned} \text{Overall efficiency} &= \frac{5833}{15000} \times 100 \\ &= 388.9\% \end{aligned}$$

$$\text{Volumetric efficiency} = \frac{\text{Actual flow rate}}{\text{Ideal flow rate}} \times 100$$

$$\begin{aligned} \text{Ideal flow rate} &= \text{nominal displacement} \times \text{speed} \\ &= 5 \times 10^{-4} \times 14.2 \\ &= 7.1 \times 10^{-3} \text{ m}^3/\text{sec} \end{aligned}$$

$$\begin{aligned} \therefore \text{Volumetric efficiency} &= \frac{5.833 \times 10^{-4}}{7.1 \times 10^{-3}} \times 100 \\ &= 8.22\% \end{aligned}$$

4) $Z = 24,000 \text{ cm} = 240 \text{ m}$

$$\text{Flow rate} = 13 \text{ litres} = 13 \text{ m}^3$$

$$\text{Since } 1000 \text{ litres} = 1 \text{ m}^3$$

$$13 \text{ litres} = \frac{13}{1000} = 0.013 \text{ m}^3/\text{sec}$$

$$\text{Velocity of a jet} = 66 \text{ m/sec}$$

$$\text{Density} = 1000 \text{ kg/m}^3$$

Using equation $P \left(P + \rho g z + \frac{\rho v^2}{2} \right) Q$

Since $P=0$ and $z=0$

$$P = \frac{\rho Q v^2}{2} = \frac{1000 \times 0.013 \times 66^2}{2}$$

$$P_{\text{out}} = 28314 \text{ W} \therefore 28.314 \text{ kW}$$

ii) At this point Density = 0 $v=0$

$$P = \rho Q g z = 1000 \times 0.013 \times 9.8 \times 240$$
$$= 30576 \text{ W}$$
$$= 30.576 \text{ kW}$$

iii) Power loss in transmission = $(30.576 - 28.314) \text{ kW}$
 $= 2.262 \text{ kW}$

Head loss in pipeline;

$$h = \frac{\text{Power transmission loss}}{\rho g Q}$$

$$= \frac{2262}{1000 \times 9.81 \times 0.013}$$

$$= 17.73 \text{ m}$$

iv) Efficiency of the pipeline and nozzle = $\frac{\text{Power of jet}}{\text{Power of reservoir}} \times 100$

$$= \frac{28314}{30576} \times 100\%$$

$$= 92.6\%$$

5) Specific gravity of Oil = 0.89

$$Z = 30,500 \text{ mm} = 30.5 \text{ m}$$

$$Q = 220 \text{ litre/sec} = 0.22 \text{ m}^3/\text{sec}$$

$$\text{Velocity of jet} = 7 \text{ m/sec}$$

(D.)
 Section 1 diameter = 300mm = 0.3m
 Section 1 Area (A_1) = $\frac{\pi \times 0.3^2}{4} = 0.071\text{m}^2$

Section 2 diameter (D_2) = 150mm = 0.15m
 " 2 Area (A_2) = $\frac{\pi \times 0.15^2}{4} = 0.018\text{m}^2$

$Q = 40 \text{ liter/sec} = 0.04\text{m}^3/\text{sec}$; $Z_1 = 10\text{m}$, $Z_2 = 6\text{m}$, $P_1 = 400\text{kN/m}^2 = 400\text{kPa}$
 $P_2 = ?$, $V_1 = \frac{Q}{A_1} = \frac{0.04}{0.071} = 0.563\text{m/s}$

$V_2 = \frac{Q}{A_2} = \frac{0.04}{0.018} = 2.27\text{m/s}$

Using Bernoulli's equation: $\frac{P_1}{\rho} + \frac{V_1^2}{2g} + Z_1 = \frac{P_2}{\rho} + \frac{V_2^2}{2g} + Z_2$

$= \frac{400}{9.81} + \frac{0.563^2}{2 \times 9.81} + 10 = \frac{P_2}{9.81} + \frac{2.27^2}{2 \times 9.81} + 6$

$\frac{P_2}{9.81} = 44.53$

$P_2 = 9.81 \times 44.53 = 436.82\text{kN/m}^2$

10) $h_{is} = 12\text{m}$ ^{depth from level} $y = 170\text{mm} = 0.17\text{m}$ of mercury; Sp. gravity of Hg = 13.6

Sp. gravity of water = 1.026

$h = y \left(\frac{S_H}{S_L} - 1 \right)$

$h = 0.17 \left(\frac{13.6}{1.026} - 1 \right)$

$h = 2.08\text{m}$

\therefore Velocity of Submarine = $\sqrt{2gh}$
 $= \sqrt{2 \times 9.81 \times 2.08}$
 $= 6.39\text{m/s}$

$$\begin{aligned} \text{Throat diameter} &= 0.076 \text{ m} \\ \text{Throat Area} &= \frac{\pi \times (0.076)^2}{4} \end{aligned}$$

$$= 4.54 \times 10^{-3} \text{ m}^2$$

$$\text{Relative density} = 0.8 \quad ; \quad \text{Pipe diameter} = 0.152 \text{ m}$$

$$\text{Pipe Area} = 0.0181 \text{ m}^2$$

$$\text{Difference between inlet and throat} = 0.919 \text{ m}$$

$$C_d = 0.97$$

$$h = \left(\frac{P_1}{\rho} - \frac{P_2}{\rho} \right) + (z_1 - z_2)$$

$$a) \text{ when } P_1 = P_2 \quad ; \quad h = 0$$

$$\text{Discharge (Q)} = C_d \times \frac{A_1 A_2}{\sqrt{A_1^2 - A_2^2}} \times \sqrt{2gh}$$

$$\text{and since } h = 0 \quad ; \quad Q = 0$$

$$b) \text{ when } P_1 - P_2 = 15170$$

$$\text{Density of liquid} = 0.8 \times 1000 = 800 \text{ kg/m}^3$$

$$h = \frac{15170}{\rho} \quad \rho = \rho g = 800 \times 9.81 = 7848 \text{ N/m}^2$$

$$h = \frac{15170}{7848}$$

$$= 1.933 \text{ m}$$

$$h = 1.933 \text{ m} + 0.919 \text{ m}$$

$$\therefore h = 2.847 \text{ m}$$

$$Q = C_d \times \frac{A_1 A_2}{\sqrt{A_1^2 - A_2^2}} \times \sqrt{2gh}$$

$$Q = \frac{0.97 \times 0.0181 \times 4.54 \times 10^{-3}}{\sqrt{(0.0181)^2 - (4.54 \times 10^{-3})^2}} \times \sqrt{2 \times 9.81 \times 2.847}$$

$$Q = 0.039 \text{ m}^3/\text{sec}$$

$$\begin{aligned}
 \text{iv) Efficiency} &= \frac{\text{Power of jet}}{\text{Power of reservoir}} \times 100 \\
 &= \frac{4797.1}{576239.4} \times 100 \\
 &= 0.83\%
 \end{aligned}$$

$$\begin{aligned}
 \text{v) Power} &= \text{Pressure} \times \text{flow rate} \\
 \text{Pressure of water} &= \rho g h = 1000 \times 9.81 \times 20 \\
 &= 196200 \text{ N/m}^2 \\
 \text{Area} &= \pi (0.05)^2 = 7.854 \times 10^{-3} \text{ m}^2 \\
 \text{Volume} &= 7.854 \times 10^{-3} \times 20 = 0.1571 \text{ m}^3 \\
 Q = \text{flow rate} &= \frac{0.1571}{60} = 2.62 \text{ m}^3/\text{sec}
 \end{aligned}$$

$$\begin{aligned}
 \text{Power} &= 196200 \times 2.62 \\
 &= 514044 \text{ W} = 514.044 \text{ kW}
 \end{aligned}$$

$$\text{vi) Inlet diameter } (D_1) = 0.3 \text{ m} \Rightarrow \frac{\pi \times 0.3^2}{4} = 0.071 \text{ m}^2$$

$$\text{Throat diameter } (D_2) = 0.2 \text{ m} \Rightarrow \frac{\pi \times 0.2^2}{4} = 0.031 \text{ m}^2$$

$$\text{CD} = 0.96, h = 0.06 \text{ m}, \text{ specific gravity of water} = 1$$

$$\text{Specific weight of gw} = 19.62 \text{ N/m}^3$$

$$\text{Specific gravity of gw} = \frac{19.62}{9.81} = 2.0$$

$$Q = \text{CD} \times \frac{A_1 A_2}{\sqrt{A_1^2 - A_2^2}} \times \sqrt{2gh}$$

$$Q = 0.96 \times \frac{0.071 \times 0.031}{\sqrt{(0.071)^2 - (0.031)^2}} \times \sqrt{2 \times 9.81 \times 0.06}$$

$$Q = 0.0359 \text{ m}^3/\text{s}$$

$$\text{Volume flowing} = 0.0359 \text{ m}^3$$

$$i) \text{ Specific gravity} = \frac{\text{Specific weight of liquid}}{\text{Specific weight of water}}$$

$$\text{S.P of liquid} = 0.89 \times 9.81$$

$$= \frac{8730.9 \text{ N/m}^3}{9.81} = 8730.9 \text{ N/m}^3$$

$$\therefore \text{density } (\rho) = \frac{8730.9}{9.81} = 890 \text{ kg/m}^3$$

$$\text{Power} = \left(P Q + \rho z Q + \frac{\rho V^2 Q}{2} \right) \rho_0$$

$$P = \frac{\rho V^2 Q}{2} = \frac{890 \times 0.22^2}{2}$$

$$= 4997.1 \text{ W} = 4.9971 \text{ kW}$$

ii) Power applied from reservoir

$$\text{Power} = \left(P Q + \rho z Q + \frac{\rho V^2 Q}{2} \right)$$

$$\text{Power} = \rho z Q$$

$$= 890 \times 300 \times 9.81 \times 0.22$$

$$= 576239.4 \text{ Watt} \approx 576.2394 \text{ kW}$$

$$iii) \text{ Power loss in transmission} = (576239.4 - 4997.1) \text{ W}$$

$$= 571442.3 \text{ Watt}$$

$$\text{Head loss} = \frac{\text{Power loss in transmission}}{\rho g Q}$$

$$= \frac{571442.3}{890 \times 9.81 \times 0.22}$$

$$= 297.5 \text{ m}$$

$$= 297.5 \text{ m}$$