

NAME: Ibeh Victor Soromtochukwu

COURSE: ENG 214 (FLUID MECHANICS)

DEPT: MECHANICAL ENGINEERING

MATRIC: 16/ENG06/029

(1) Rate of pump delivery =  $10 \text{ dm}^3/\text{min}$   
 Pressure change =  $12 \text{ bar} = 12 \times 10^5 \text{ N/m}^2$   
 speed of rotation =  $1500 \text{ rev/min} = \frac{1500 \text{ rev}}{1 \text{ min}} \times \frac{1 \text{ min}}{60 \text{ sec}} = 25 \text{ rev/sec}$   
 nominal displacement =  $10 \text{ cm}^3/\text{rev}$   
 Torque input =  $12.5 \text{ Nm}$

(i) Volumetric efficiency =  $\frac{\text{Actual flow rate}}{\text{Ideal flow rate}} \times 100\%$

Actual flow rate =  $10 \text{ dm}^3/\text{min} = \frac{10 \text{ dm}^3}{1 \text{ min}} \times \frac{1 \text{ m}^3}{1000 \text{ dm}^3} \times \frac{1 \text{ min}}{60 \text{ sec}} = 1.667 \times 10^{-4} \text{ m}^3/\text{sec}$

nominal displacement =  $\frac{10 \text{ cm}^3}{1 \text{ rev}} \times \frac{1 \text{ m}^3}{1000 \text{ cm}^3} = 1 \times 10^{-5} \text{ m}^3/\text{rev}$

Ideal flow rate = nominal displacement  $\times$  speed  
 $= 1 \times 10^{-5} \times 25 = 2.5 \times 10^{-4} \text{ m}^3/\text{sec}$

volumetric efficiency =  $\frac{1.667 \times 10^{-4}}{2.5 \times 10^{-4}} \times 100 = 66.68\%$

(ii) fluid power = Actual flow rate  $\times$  pressure  
 $= 1.667 \times 10^{-4} \times 12 \times 10^5 = 200.04 \text{ watts}$

(iii) shaft power = Torque input  $\times$  angular speed  
 Torque input =  $12.5 \text{ Nm}$

Angular speed =  $\omega = \frac{2\pi N}{60} = \frac{2 \times \pi \times 1500}{60} \text{ (rps)}$

$\therefore$  shaft power =  $12.5 \times 2 \times \pi \times 25 = 1963.5 \text{ watts}$

(iii) Overall efficiency =  $\frac{\text{fluid power}}{\text{shaft power}} \times 100\%$

$$= \frac{200.4}{1968.5} \times 100$$

$$= 10.21\%$$

- (2) Rate of delivery =  $35 \text{ dm}^3/\text{min} = 35/60 = 0.58 \text{ dm}^3/\text{sec}$   
 Pressure change =  $100 \text{ bar} = 100 \times 10^5 \text{ N/m}^2$   
 Overall efficiency =  $87\%$   
 Shaft power = ?

$$\text{Rate of delivery} = 0.58 \times \frac{1}{1000} = 5.83 \times 10^{-4} \text{ m}^3/\text{sec}$$

$$\text{Fluid power} = \text{Actual flow rate} \times \text{pressure change}$$

$$= 5.83 \times 10^{-4} \times 100 \times 10^5$$

$$= 5833.33 \text{ watts}$$

$$\text{Overall efficiency} = \frac{\text{Fluid power}}{\text{Shaft power}} \times 100$$

$$87 = \frac{5833.33 \times 100}{\text{S.P}}$$

$$\frac{0.87}{5833.33} = \frac{1}{\text{S.P}}$$

$$\therefore \text{Shaft power} = 6704.977 \text{ watts.} //$$

- (3) Nominal displacement =  $50 \text{ cm}^3/\text{rev} = \frac{50 \text{ cm}^3}{\text{rev}} \times \frac{1 \text{ m}^3}{1000^3 \text{ cm}^3} = 5 \times 10^{-4} \text{ m}^3/\text{rev}$

$$\text{Pressure change} = 100 \text{ bar} = 100 \times 10^5 \text{ N/m}^2$$

$$\text{Shaft power} = 15 \text{ kW} = 15,000 \text{ W}$$

$$\text{Actual flow rate} = 35 \text{ dm}^3/\text{min} = \frac{35 \text{ dm}^3}{1 \text{ min}} \times \frac{1 \text{ min}}{60 \text{ sec}} \times \frac{1 \text{ m}^3}{1000 \text{ dm}^3}$$

$$= 5.833 \times 10^{-4} \text{ m}^3/\text{sec}$$

$$\text{Speed of rotation} = 850 \text{ rpm} = 14.2 \text{ rps} //$$

$$\text{Overall efficiency} = \frac{\text{Actual fluid power}}{\text{Shaft power}} \times 100$$

$$\begin{aligned}\text{Fluid power} &= \text{Actual flow rate} \times \text{pressure change} \\ &= 5.833 \times 10^{-4} \times 100 \times 10^5 \\ &= 5833 \text{ watts}\end{aligned}$$

$$\begin{aligned}\text{Shaft power} &= \text{Torque input} \times \text{angular speed} \\ &= 15 \text{ kW} = 15000 \text{ W}\end{aligned}$$

$$\begin{aligned}\text{Overall efficiency} &= \frac{5833 \times 100}{15000} \\ &= 388.9\% \quad //\end{aligned}$$

$$\text{Volumetric efficiency} = \frac{\text{Actual flow rate}}{\text{Ideal flow rate}} \times 100$$

$$\begin{aligned}\text{Ideal flow rate} &= \text{nominal displacement} \times \text{speed} \\ &= 5 \times 10^{-4} \times 14.2 \\ &= 7.1 \times 10^{-3} \text{ m}^3/\text{sec}\end{aligned}$$

$$\begin{aligned}\text{Volumetric efficiency} &= \frac{5.833 \times 10^{-4} \times 100}{7.1 \times 10^{-3}} \\ &= 8.22\%\end{aligned}$$

$$(4) \quad z = 24,500 \text{ cm} = \frac{24,500}{100} = 245 \text{ m}$$

$$\text{Flow rate} = 13 \text{ litres/sec}$$

$$\text{Since } 1000 \text{ litres} = 1 \text{ m}^3$$

$$\therefore 13 \text{ litres} = \frac{13 \times 1}{1000} = 0.013 \text{ m}^3/\text{sec}$$

$$\text{Velocity of jet} = 66 \text{ m/sec}$$

Jet issuing from nozzle is at atmospheric pressure and at datum level.

$$p = 0 ; z = 0$$

$$\text{Density} = 1000 \text{ kg/m}^3$$



from equation;  $P = \left( p + \rho g z + \frac{\rho v^2}{2} \right) Q$

Since

$$p = 0; z = 0$$

$$P = 0 \cdot Q + \frac{\rho Q v^2}{2} + \rho g \cdot 0$$

$$P = \frac{\rho Q v^2}{2} = \frac{1000 \times 0.013 \times 66^2}{2}$$

$$P = 28314 \text{ W} = 28.314 \text{ kW}$$

(ii) At this point;  $p = 0$  while  $v = 0$

$$P = \left( p + \rho g z + \frac{\rho Q v^2}{2} \right)$$

$$P = \rho g z = 1000 \times 0.013 \times 9.8 \times 240$$

$$= 30576 \text{ W}$$

$$= 30.576 \text{ kW}$$

(iii) Power loss in transmission =  $30.576 - 28.314$   
 $= 2.262 \text{ kW} = 2262 \text{ W}$

Head loss in pipeline;

$$h = \frac{\text{Power transmission loss}}{\rho g Q}$$

$$= \frac{2262}{1000 \times 9.81 \times 0.013} = 17.73 \text{ m}$$

(iv) Efficiency of the pipeline and nozzle;

$$\Rightarrow \frac{\text{Power of jet}}{\text{Power of reservoir}} \times 100 = \frac{28314}{30576} \times 100$$

$$\Rightarrow 92.6\%$$

(5) specific gravity of oil = 0.89

$z = 30,000 \text{ cm} = 300 \text{ m}$

$Q = 220 \text{ litres/sec}$

Since  $1000 \text{ litres} = 1 \text{ m}^3$

$220 \text{ litres} = \frac{220 \times 1}{1000} = 0.22 \text{ m}^3/\text{sec}$

velocity of jet =  $7 \text{ m/sec}$ .

(i) specific gravity =  $\frac{\text{specific weight of liquid}}{\text{specific weight of water}}$

$0.89 \times 9.81 = \text{specific weight of liquid}$

$\Rightarrow 8.7309 \text{ kN/m}^3 = 8730.9 \text{ N/m}^3$

$\therefore \text{density } (\rho) = \frac{8730.9}{9.81} = 890 \text{ kg/m}^3$

Recall:

At the point of jet issuing from nozzle;  $P=0$ ;  $z=0$   
from equation;  $(PQ + \rho g z Q + \frac{\rho V^2 Q}{2}) = \text{Power}$

$P = \frac{\rho V^2 Q}{2} = \frac{890 \times 7^2 \times 0.22}{2}$

$= 4797.1 \text{ W} = 4.7971 \text{ kW}$

(ii) Power supplied from reservoir,  $P=0$ ;  $V=0$

$\text{Power} = (PQ + \rho g z Q + \frac{\rho V^2 Q}{2})$

$\text{Power} = \rho g z Q = 890 \times 300 \times 9.81 \times 0.22$

$= 576239.4 \text{ W} = 576.2394 \text{ kW}$

(iii) Power loss in transmission =  $576239.4 - 4797.1$   
 $= 571442.3 \text{ W}$

Head used =  $\frac{\text{Power loss in transmission}}{\rho g Q}$

$$\Rightarrow \frac{571442.3}{890 \times 9.81 \times 0.22}$$

$$= 297.50 \text{ m}$$

(v) efficiency =  $\frac{\text{power of jet}}{\text{power of reservoir}} \times 100$

$$= \frac{4797.1}{576239.4} \times 100$$

$$= 0.83\%$$

(6) Power = pressure  $\times$  flow rate

$$\text{pressure of water} = \rho gh = 1000 \times 9.81 \times 20$$

$$= 196200 \text{ N/m}^2$$

$$\text{Area} = \pi \times (0.05)^2 = 7.854 \times 10^{-3} \text{ m}^2$$

$$\text{Volume} = 7.854 \times 10^{-3} \times 20 = 0.1571 \text{ m}^3$$

$$Q = \text{flow rate} = \frac{0.1571}{60} = 2.62 \text{ m}^3/\text{s}$$

$$\text{power} = 196200 \times 2.62 = 514044 \text{ W} = 514.044 \text{ kW}$$

(7) Inlet diameter =  $0.3 \text{ m} \Rightarrow \frac{\pi \times 0.3^2}{4} = 0.071 \text{ m}^2 = \text{Inlet Area } (A_1)$

Throat diameter =  $0.2 \text{ m} \Rightarrow \frac{\pi \times 0.2^2}{4} = 0.031 \text{ m}^2 = \text{Throat Area } (A_2)$

$$C_d = 0.96 ; h = 0.06 \text{ m} ; \text{Sp. gr. of mercury} = 13.6$$

$$\text{Sp. gr. of water} = 1$$

$$\text{Sp. gr. of gas}$$

$$\text{Sp. gr. of gas} = 19.62$$

$$\text{Specific weight of gas} = 19.62 \text{ N/m}^3$$

$$\text{Sp. gr. of gas} = \frac{19.62 \div 1000}{9.81} = 0.002$$

$$Q = C_d \times \frac{A_1 A_2}{\sqrt{A_1^2 - A_2^2}} \times \sqrt{2gh}$$



$$Q = 0.96 \times 0.071 \times 0.031 \times \frac{\sqrt{2 \times 9.81 \times 0.06}}{\sqrt{(0.071)^2 - (0.031)^2}}$$

$$Q = 0.0359 \text{ m}^3/\text{s}$$

$$\text{Volume flowing} = 0.0359 \text{ m}^3$$

⑧ Throat diameter ( $D_2$ ) = 0.076 m

$$\text{Throat area } (A_2) = \frac{\pi \times (0.076)^2}{4} = 4.54 \times 10^{-3} \text{ m}^2$$

Relative density = 0.8

Pipe diameter = 0.152 m =  $D_1$

Pipe Area ( $A_1$ ) = 0.0181 m<sup>2</sup>

Difference between inlet and throat = 0.914 m

$C_d = 0.97$

$$\text{Since } h = \left( \frac{P_1}{\rho} - \frac{P_2}{\rho} \right) + (z_1 - z_2)$$

(a) When  $P_1 = P_2$

$$\therefore h = (z_1 - z_2) \therefore h = 0$$

$$\text{Discharge } (Q) = C_d \times \frac{A_1 A_2}{\sqrt{A_1^2 - A_2^2}} \times \sqrt{2gh}$$

Since  $h = 0$

$$\therefore Q = 0$$

(b) when  $P_1 - P_2 = 15170$ ;  $0.8 \times 1000 = 800 \text{ kg/m}^3 = \text{density of liquid}$

$$h = \frac{15170}{7848}$$

$$7848$$

$$W = \rho g$$

$$W = 800 \times 9.81 = 7.848 \text{ kN/m}^3$$

$$h = 1.933 \text{ m} + 0.914 \text{ m}$$

$$\therefore h = 2.847 \text{ m}$$

$$\therefore \text{Discharge } (Q) = C_d \times \frac{A_1 A_2}{\sqrt{A_1^2 - A_2^2}} \times \sqrt{2gh}$$

$$Q = 0.97 \times \frac{0.0181 \times 4.54 \times 10^{-3}}{\sqrt{(0.0181)^2 - (4.54 \times 10^{-3})^2}} \times \sqrt{2 \times 9.81 \times 2.847}$$

$$Q = 0.034 \text{ m}^3/\text{seconds}$$

(9) section 1 diameter = 300 mm = 0.3 m ( $D_1$ )  
 section 1 Area =  $\frac{\pi \times 0.3^2}{4} = 0.071 \text{ m}^2 (A_1)$

section 2 diameter = 150 mm = 0.15 m ( $D_2$ )  
 section 2 Area =  $\frac{\pi \times 0.15^2}{4} = 0.018 \text{ m}^2 (A_2)$

$Q = 40 \text{ litres/sec} = 0.04 \text{ m}^3/\text{sec}$

$z_1 = 10 \text{ m}, z_2 = 6 \text{ m}$

$P_1 = 400 \text{ kN/m}^2 \quad P_2 = ?$   
 $= 400 \text{ kPa}$

$V_1 = \frac{Q}{A_1} = \frac{0.04}{0.071} = 0.563 \text{ m/s}$

$V_2 = \frac{Q}{A_2} = \frac{0.04}{0.018} = 2.27 \text{ m/s}$

from Bernoulli's equation:  $\frac{P}{\rho} + \frac{V^2}{2g} + z = \frac{P_2}{\rho} + \frac{V_2^2}{2g} + z_2$

$\frac{400}{9.81} + \frac{0.563^2}{2 \times 9.81} + 10 = \frac{P_2}{9.81} + \frac{2.27^2}{2 \times 9.81} + 6$

$\frac{P_2}{9.81} = 44.53$

$\therefore$  Intensity of pressure at section 2  $\Rightarrow 9.81 \times 44.53$

$P_2 = 436.82 \text{ kN/m}^2$

(10) axis = 12 m below sea-level.

$y = 170 \text{ mm} = 0.17 \text{ m}$  of mercury

Sp. gravity of Hg = 13.6

Sp. gravity of water = 1.026

$h = y \left( \frac{S_{H_1}}{S_1} - 1 \right)$

$h = 0.17 \left( \frac{13.6}{1.026} - 1 \right)$

$h = 2.08 \text{ m}$

$$\therefore \text{Velocity of submarine (V)} = \sqrt{2gh} = \sqrt{2 \times 9.81 \times 2.08} \\ = 6.39 \text{ m/s}$$